# Asia Pacific Research Initiative for Sustainable Energy Systems 2013 (APRISES13)

Office of Naval Research Grant Award Number N00014-14-1-0054

# CEILING FAN STUDY: LITERATURE AND MARKET REPORT

# Task 7

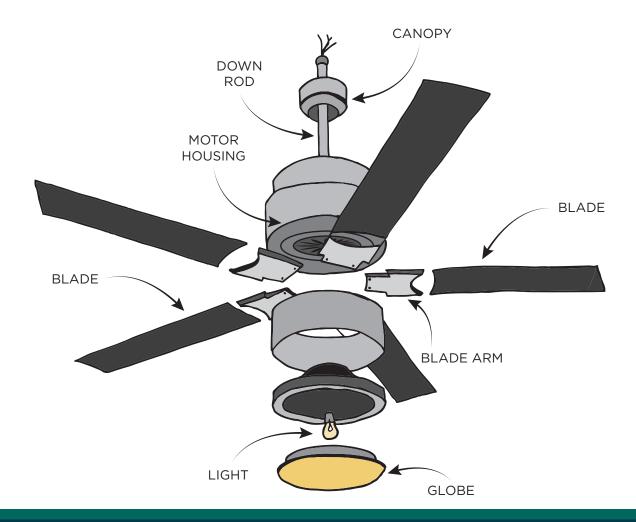
Prepared For Hawaii Natural Energy Institute

> Prepared By MKThink

August 2017







# DELIVERABLE 2: FINAL LITERATURE AND MARKET REPORT

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#### HNEI CONTRACT NO. Z10152521

University of Hawaii's Asia-Pacific Research Initiative for Sustainable Energy Systems (APRISES)

Task 7 - Energy Efficiency

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Contact: James Maskrey, MEP, MBA Project Manager maskrey2@hawaii.edu (808) 956-3645 1680 East West Road, Suite 109 Honolulu, HI 96822

#### Prepared by MKThink

Principal in Charge: Mark R. Miller, FAIA LEEDAP Research Team Lead: Amy Nagengast, PhD PE LEEDAP

Research Team: Nate Goore, Sean Dasey, Signo Uddenberg, Adeniyi Harrison, Mayssen Labidi

Partners: Roundhouse One

Contact: Signo Uddenberg, EIT LEEDAP uddenberg@mkthink.com (415) 288-3389 1500 Sansome Street San Francisco, CA. 94111

## Project Introduction

| Project Introduction | 2 |
|----------------------|---|
| Project Organization | 3 |
| Phase 1 Overview     | 4 |
| Research Methodology | 5 |



| Introduction8       |
|---------------------|
| History9            |
| Market Assessment10 |
| Fan Types12         |

# 3

| Motors                      | 16 |
|-----------------------------|----|
| Blades                      | 17 |
| Controls                    |    |
| Flat Blade Air Movement     | 25 |
| Air Foil Blade Air Movement |    |
| Modeling                    | 32 |
| Codes and Standards         | 35 |

Design

## Sizing and Placement

| Flat Blade Sizing and Placement    |
|------------------------------------|
| Flat Blade Obstructions            |
| Airfoil Blade Sizing and Placement |
| Airfoil Blade Obstructions         |
| Lighting Obstructions51            |



## Operations

| Eporav              | 51 |
|---------------------|----|
| Energy              |    |
| Performance Metrics | 56 |
| Thermal Comfort     | 60 |
| Air Movement        | 62 |
| Alliesthesia        | 67 |



## **Literature Review**

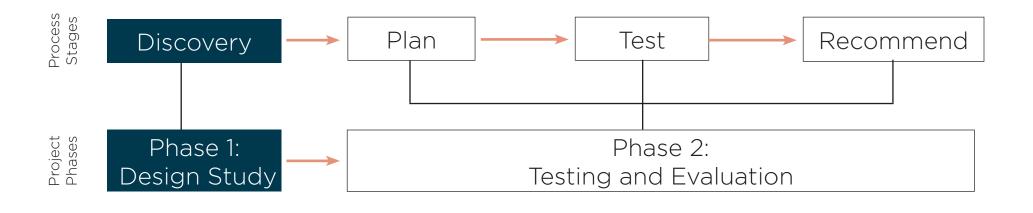
| Introduction72                         |
|--|
| Source & Content Diversity73           |
| Bibliography75                         |
| Acronyms91                             |
| Appendix: Electronic Literature Review |

# Project Introduction

| Project Introduction | 2 |
|----------------------|---|
| Project Organization | 3 |
| Phase 1 Overview     | 4 |
| Research Methodology | 5 |

Hawaii and other tropical areas seek innovative applications to improve thermal comfort. While ceiling fans are not new, no comprehensive document is available to designers and planners to make deliberate, conscious fan selection decisions based on research, space constraints and physical characteristics of fans. This project intends to distill and compile information from a variety of sources into a report in order to address this limitation. In support of the University of Hawaii's 'Asia Pacific Research Initiative for Sustainable Energy Systems (APRISES)' Task 7 Energy Efficiency, this project will evaluate fan typologies, control options, and operational characteristics and provide research-based design guidelines to assist in space planning, design and product selection. A ceiling fan for this research is defined as is done by the Code of Federal Regulations (CF) as "a non-portable device that is suspended from a ceiling for circulating air via the rotation of fan blades".

In the subsequent pages, the overall ceiling fan project organization as well as the specific objectives for Phase 1 are outlined. This ceiling fan project is organized into two phases. Phase 1 focuses on acquiring and summarizing ceiling fan knowledge from academic and industry sources while identifying gaps for further study. Phase 2 defines a research study from the Phase 1 gap analysis. Furthermore, Phase 2 involves the installation, testing and evaluation of ceiling fans at a selected site in Hawaii in support of the research study.



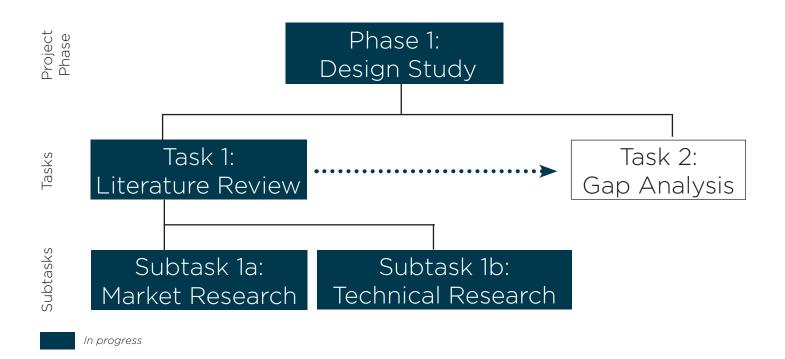
In progress



The objectives of Phase 1 are:

- Define typologies of ceiling fan applications and a portfolio of existing and emerging fan technologies
- Develop metrics structure to evaluate the performance of the portfolio of fan technologies
- Conduct literature survey of current CFD simulation techniques and their applicability of quantifying performance of selected fan typologies.

Phase 1 is separated into Task 1: Literature Review and Task 2: Gap Analysis. The purpose of Task 1 is to investigate available literature to understand the current state of the ceiling fan industry. Task 2 will use the knowledge gleaned in Task 1 to determine gaps in knowledge and research. This report only examines Task 1 which is comprised of Subtask 1a: Market Research and Subtask 1b: Technical Research, and both are discussed on the following page.





The goal of Task 1 was to examine the available literature to understand the current state of the ceiling fan industry in the United States. While the ceiling fan industry is ever expanding and changing, this research focused on two key areas. Subtask 1a analyzed manufacturer and industry organization information to understand current design features (e.g. motors, blades, controls) and performance characteristics (e.g. placement and sizing guidelines, comparison metrics) of ceiling fans. Information was obtained through sources such as conference artifacts, technical specifications, newsletters, and phone conversations. Subtask 1b summarized academic and governmental research to better identify future emerging trends and technologies. In particular, this effort focused on the impact of air movement on occupant comfort and satisfaction. Information was obtained through peer-reviewed articles, governmental reports and phone conversations with researchers.

At a high level, the approach began with a focused look at each Subtask 1a & 1b across the contract specific categories (e.g. design, placement, energy, thermal comfort) using a source hierarchy. The source hierarchy prioritized credible sources such as government and academic research over personal blogs. This approach ensured both breadth and guality of information. Each source often had multiple articles on different aspects of ceiling fans. In order for guick access and recall, each article was recorded into a matrix with key information extracted related to the contract categories. In practice, content crossed subtask boundaries. For example, air movement was discussed in ceiling fan design (Subtask 1a) and also how air movement impacted occupants (Subtask 1b). Therefore the following literature review is organized around content areas which have a mixture of Subtask 1a & 1b sources and articles.



Ceiling Fan Overview

| Ceiling Fan Design Study Introduction |    |
|---------------------------------------|----|
| Ceiling Fan History                   | 9  |
| US Ceiling Fan Market                 | 1C |
| Fan Types                             | 12 |

The following Ceiling Fan report is organized around five primary sections namely Ceiling Fan Overview, Design, Sizing and Placement, Operations and Literature Review based on information gathered through available industry and academic literature supplemented with phone conversations with relevant industry professionals.

The Ceiling Fan Overview looks into the history, market characteristics and main types of ceiling fans.

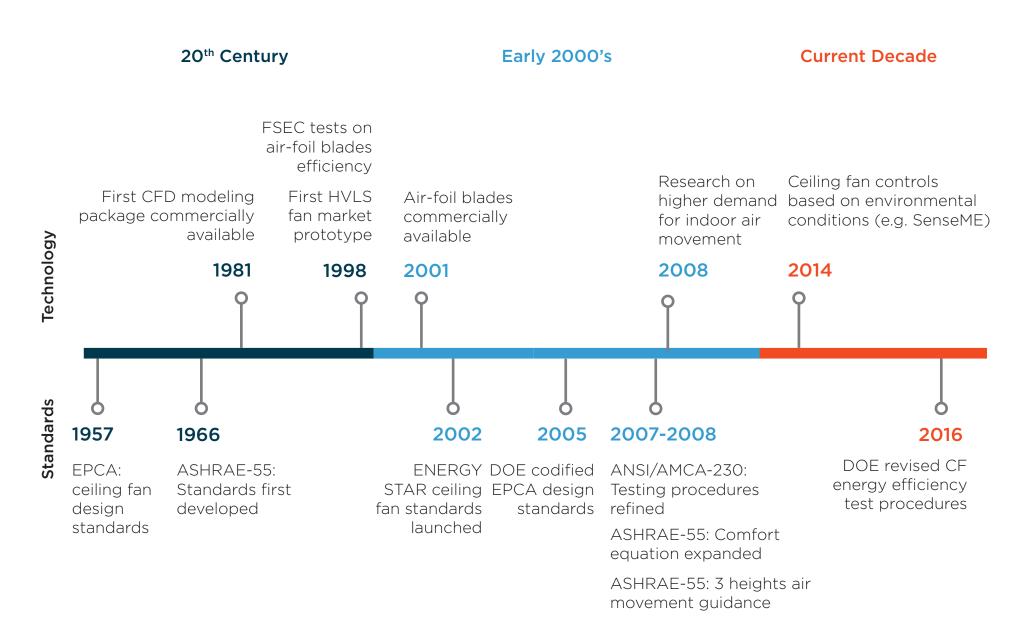
The Design section introduces key hardware components (i.e. blades, motors, and controls) and generalized air movement profiles of flat blades and airfoil blades. Also, ceiling fan modeling is included as a design tool that helps professionals understand room air temperature and speed distribution.

The Sizing and Placement section discusses single and multiple ceiling fan sizing guidelines related to room size (width x length) and placement with respect to horizontal and vertical room obstructions.

The Operations section outlines energy consumption and efficacy information to help estimate operational costs. Also, air speed, direction and intermittency are explored in their relation to occupant thermal comfort.

The Literature Review section categorizes the sources which have informed the content in the previous sections.

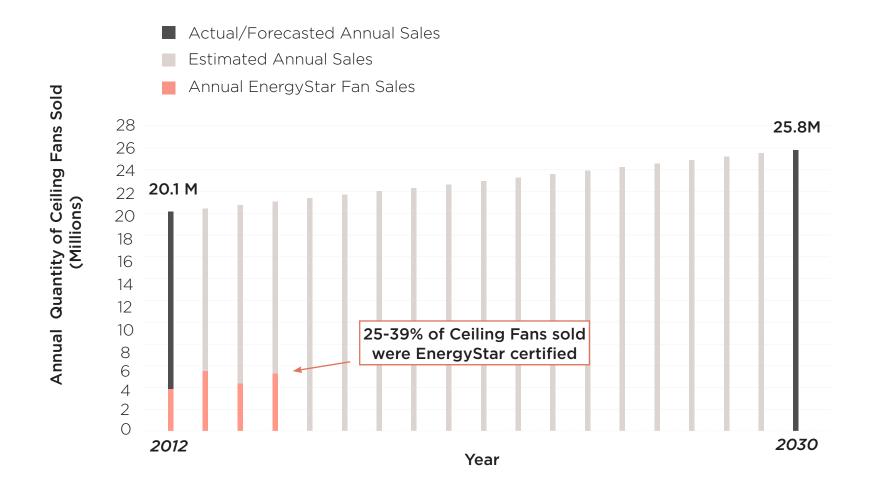




Sources: Paliaga, 2017. MacroAir Fans, 2017. Ullman, 2013. ASHRAE, n.d., Big Ass Solutions, 2014 (a), Gossamer Wind, n.d, ENERGY STAR, 2016, Symscape, 2008



The United States Ceiling Fan Market is steadily growing. Although there is little information available on market trends, key player distributions and competitive landscapes, shipment data and sales forecasts are used below to arrive at actual and forecasted sales values. The annual quantity of ceiling fans sold in 2012 was 20.1 million and is predicted to reach 25.8 million in 2030. The market share of ENERGY STAR rated ceiling fans also grew with a penetration of 25-39% between 2012-2015.



Sources: Sathaye, et al, 2012. Energy Star, 2012-2015

Periodically, the U.S. Department of Energy (DOE) reviews energy efficiency standards for consumer products and commercial or industrial equipment such as ceiling fans in order to determine if energy efficiency standards need to be updated and if so, if these improvements are technologically viable and economically justifiable. To better understand the impact of more stringent standards on ceiling fan consumers, the DOE conducts estimations of the existing market share distribution by common blade size for different fan types. To do so, data is acquired online and through in-person discussions with large fan retailers, and then validated by retailers, such as Westinghouse Lighting in a public review. The below information is the blade span market distribution for standard and largediameter fans that are greater than 7 feet (HVLS ceiling fans) for the year 2015 in the no-new standards scenario with no efficiency standards improvements. These two types of ceiling fans are defined in the following page.

|                          | Standard Ceiling Fan |       |      | Large-Diameter Ceiling Fan (HVLS) |       |       |
|--------------------------|----------------------|-------|------|-----------------------------------|-------|-------|
| Fan Diameter<br>(inches) | 44"                  | 52"   | 60"  | 96"                               | 144"  | 240"  |
| Market Share<br>(%)      | 21.1%                | 72.5% | 6.5% | 22.0%                             | 27.0% | 51.0% |

#### 2015 Market Share Distribution by Blade Span

Sources: U.S. Department of Energy, 2016 (b). Paliaga, 2017

Several types of ceiling fans exist and are emerging to satisfy evolving demands for energy-efficient ventilation and improved air movement distribution. Each fan type differs in either functionality, design or performance and the applications vary from residential, commercial and industrial. To compare each fan type's performance, CFM (cubic feet per minute) is used to determine the amount of airflow per minute with respect to each fan type's average diameter size or range. For fan's larger than 7 feet in diameter, is common to see either volumetric (CFM) or velocity metrics (fpm (feet per minute)) represented.

| Standard Technology   Emerging Technology |   |   |  |   |
|---|---|---|--|---|
|   | Standard Fan  | Dual-Head Fan   | HVLS Fan<br>(High-Volume Low Speed)  | Smart Ceiling Fan   |
| Definition                                | Conventional fans are fans<br>with diameters that are<br>equal to or less than 7 feet<br>(84"), whose lowest point<br>on the blade is more than 10<br>inches from the ceiling | Dual-head fans have two<br>opposing head fixtures that<br>direct air in the direction<br>they face, allowing for more<br>targeted air motion and<br>enhanced spread | HVLS fans are fans with<br>airfoil blades and diameters<br>greater than 7 feet (84")<br>that operate at low speeds<br>and distribute large volumes<br>of air | Smart ceiling fans are fans<br>that automatically adjust fan<br>settings and speed levels<br>based on a room's occupant<br>activity and environmental<br>conditions |
| чо  | 42" - 49" (3.5 - 4 ft)<br>diameter  | (2) 44" - 52" (3.6 - 4.3 ft)<br>diameter  | 192" (16 ft) diameter  | 52" - 84" (4.3 - 7 ft)<br>diameter  |
| Avg Configuration                         |   |   |  |   |
| Avg<br>CFM                                | 1,999 - 5,000 CFM   | 6,300 - 8,300 CFM   | 166,000 CFM  | ~ 5,300 - 25,000 CFM  |

Sources: E3T: Energy Efficiency Emerging Technologies, n.d. U.S. Department of Energy, 2016 (a). U.S. Department of Energy, 1989. Wayfair, n.d. Hollist, 2017. MacroAir Fans, 2016. Leading Edge: Marley Engineered Products, 1999. CeilingFan.Org, 2000. Hansen Wholesale, n,d (a). Blue Giant, 2007 Image Sources: Gromol Niy, 2014, Pinterest, n.d., Arcat, n.d., MakeUseOf, n.d

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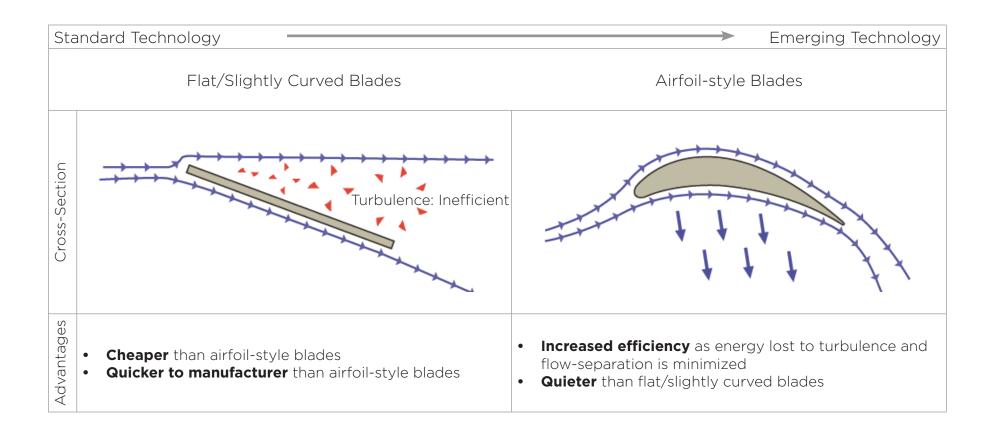
# Design

| Motors                     | 16 |
|----------------------------|----|
| Blades                     | 17 |
| Controls                   | 20 |
| Flat Blade Air Movement    | 25 |
| Airfoil Blade Air Movement | 26 |
| Modeling                   | 32 |
| Codes & Standards          | 35 |

Motor efficiency increases from Alternating Current (AC) Induction motors to Brushless Direct Current (DC) motors. The motor types differ in energy efficiency, cost and durability. The table below shows the technologies for motors, ranging from the standard and traditional components to the emerging ones.

| Stan            | dard Technology ————   |  | Emerging Technology  |
|-----------------|--|--|--|
|                 | AC Induction Motors  | Permanent Magnet DC<br>(PMDC) Motors   | Brushless DC Motors  |
| How It<br>Works | The moving magnetic field causes rotor movement or rotation  | The segmented commutator rotates<br>within a stationary magnetic field<br>which produces mechanical switching<br>of the armature current                                     | The magnetic field rotates and<br>commutation occurs when the stator<br>current direction is switched at precise<br>intervals in relation to the rotating<br>magnetic field          |
| Cross-Section   | Windings are in stator assembly and there<br>is a squirrel cage rotor<br>Stator<br>Windings Kator<br>Windings Rotor                  | Windings are in rotor assembly and<br>permanent magnets are for stator<br>assembly<br>Steel<br>Ring<br>Wound<br>Armature<br>Permanent Magnet<br>bonded to Steel Ring         | Rotor has permanent magnets and stator<br>has windings<br>Stator<br>Fermanent Magnet<br>bonded to Rotor  |
| Advantages      | <ul> <li>Constant, even airflow provided as opposed to variable, uneven flow from DC fans</li> <li>Cheaper than DC motors</li> </ul> | <ul> <li>Higher horsepower per pound and dollar, than any AC fan</li> <li>Flat torque over wide speed range, while AC motors often lose torque as speed increases</li> </ul> | <ul> <li>Most energy efficient, consuming<br/>70% less energy than AC motors</li> <li>Quietest, due to lack of friction</li> <li>Longer service life than PMDC<br/>motors</li> </ul> |

Sources: Galco Industrial Electronics, n.d., Bodine Electric Company, 2003. Pelonis, 2014. Manley, 2013. Henley Fan, 2013. Sawyers, 2011 Image Source: Orientalmotor, n.d Blade shape has evolved from flat or slightly curved blades to airfoil blades to increase efficiency. Optimal blade design requires a balance between maximizing air speed, creating uniform air speed along the fan radius and maximizing airflow coverage. The table below compares flat or slightly curved blades with airfoil-style blades commonly found on HVLS fans.





220

120

100

80

60

40

20

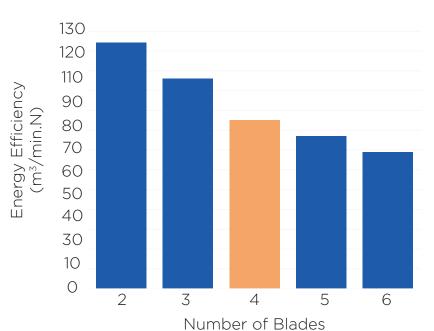
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The number of blades on a ceiling fan is important to its performance. While additional blades create more airflow, fan weight and drag also increase, which requires the fan motor to work harder. Studies show that the airflow behavior improves by less than 10% when the number of blades exceeds four flat blades.

Key Takeaway: Four ceiling fan blades are often used in residential buildings to optimize for both airflow

and energy efficiency

### **Comparison of Energy Efficiency and Blade Quantity**



## 200 Volumetric Flow Rate (m<sup>3</sup>/min) 180 160 140

3

4

Number of Blades

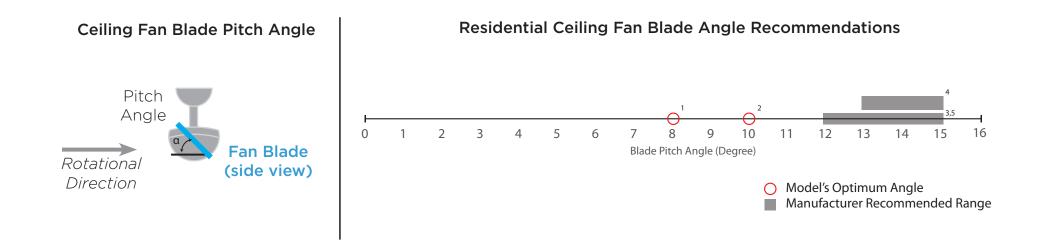
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6

**Comparison of Airflow and Blade Quantity** 

## Sources: Adeeb, et al, 2016. Adeeb, et al, 2015

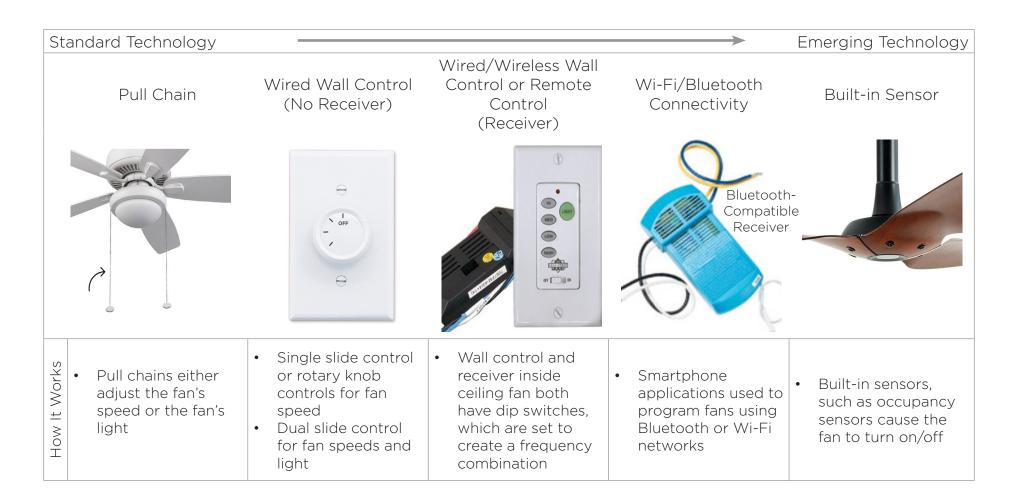
A ceiling fan's blade pitch is the angle between a horizontal plane and the blade tilt that assists in generating airflow. As this angle becomes steeper, more airflow is created, however this requires the motor to progressively work harder to maintain the same speeds. For residential fans, academic modeling studies have found the optimal blade pitch to be 8-10° while manufacturers recommend 12-15°.



Sources: Swaroop, 2017. Sali, 2016. Lumen Light and Living, 2017. CeilingFans.com, 2017. Littman Bros Lighting, 2017 Image Source: Singh, 2014



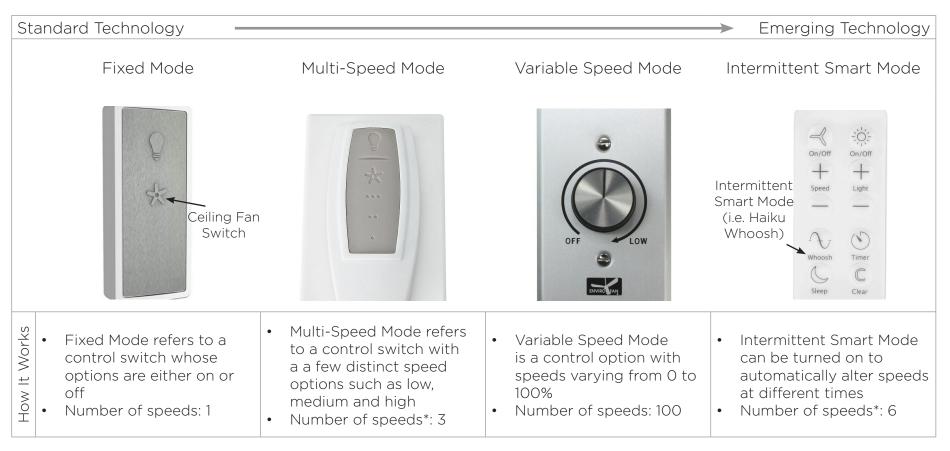
The table below shows the technologies for ceiling fan controls, ranging from the standard and traditional technologies to emerging technologies. The ceiling fan control types are Pull Chain, Wired Wall Control, Wireless Wall Control/Remote Control, Wifi/Bluetooth Connectivity and Smart Mode. These control types have different features, such as the number of speeds and fans that can be controlled, all of which is discussed in subsequent pages.



Sources: ENERGY STAR, 2016, Bermudez, 2014 Image Sources: Lowes, n.d., Tanger Ecoles, n.d. Hunter Fan, n.d. New Atlas, 2014



Various control modes exist and are emerging within ceiling fan control devices. These control modes are either Fixed Mode, Multi-Speed Mode, Variable Speed Mode or Continuous Smart Mode. Each control mode differs in the number of speeds that it can control, and newer technologies such as the Continuous Smart Mode are also able to automatically alter speeds at different times throughout the day to simulate natural breezes.



\* The number of speeds noted for Multi-Speed Mode and Intermittent Smart Mode are specific to the devices shown above, and may vary by control model. For the Intermittent Smart Mode, the number of speeds refer to the speeds that are available on the device that the fan will adjust to at irregular intervals once the Whoosh Mode is turned on. In this scenario, the fan will adjust to either of the six available speeds.

Sources: Home Depot, n.d. (b), Home Depot, n.d. (c), Acer Trading Group, 2017, Haiku Home, 2017, Haiku Home, 2015 Image Sources: Home Depot, n.d. (b), Home Depot, n.d. (c), Acer Trading Group, n.d., Haiku Home, 2017



Several options exist for controlling multiple ceiling fans with one control technology. Existing ceiling fans can integrate wall, remote and smart-control systems by installing new electrical equipment. In comparison, built-in occupancy sensor controls are embedded into an existing fan or control system, therefore acquiring sensor features requires the purchase of a new ceiling fan. Below are brief explanations on how to incorporate multiple ceiling fans into the different control types.

**Wall Control:** The amperage of the wall control unit is the determining factor for figuring out the number of ceiling fans which can be simultaneously controlled. As an example, considering that one ceiling fan draws close to 1 amp, a wall control unit that has a load capacity of 5 amps would only be able to control about 4-5 ceiling fans simultaneously. Different types of wall control devices have different load capacities, and the maximum load capacity that can be found on a device is 15-20 amps.

### Existing Ceiling Fan?

**Remote Control:** Linking multiple ceiling fans to a single, individual remote control involves purchasing additional receivers per additional ceiling fan, and then resetting frequency settings such that the dip-switches in the receiver and the remote control match.

**Smart-Control:** Like the remote control pairing process, linking multiple ceiling fans to a Wi-Fi/ Bluetooth Plug-In Controller control requires purchasing an additional Bluetooth compatible receiver per additional fan and installing it within the fan's canopy. Some Bluetooth Plug-In Controllers, however, only work within a certain range, for instance less than 80 feet.

**New Ceiling Fan? Built-In Sensor:** Smart occupancy sensors are enabled either within the ceiling fan or in the wallmounted control, however these cannot be added to an existing fan as they come embedded in a new system. The control of multiple ceiling fans therefore is not a result of the occupancy sensor control additive, but rather the capacity load of the wall control it is originally embedded into.

Sources: Hunter Fan, 2015., Envirofan, n.d. (b), Home Depot, 2015., Hunter Fan, 2013., SkyBlade Fans, 2017 (a), SkyBlade Fans, 2017 (b)., Big Ass Solutions, n.d. (b), Big Ass Solutions, 2017, Lutron, 2017, Crestron, 2017, Hunter Fan, 2017

Ceiling fan control technologies vary in physical design and functionality, therefore when selecting for a multiple fan control device, control options as well as speed and lighting features need to be considered. The number of fans that can be simultaneously controlled depends on fan size, type and model and so the range of values that can be found in the below table are aggregations of several existing products. In other words, the maximum values of fans and speeds that can be controlled does not apply to all available products, rather it is simply the upper range found in the industry.

| Control Type  | Control<br>Location   | Personal<br>Control | Number of fans controlled          | Number of<br>Speeds        | Lighting<br>Controls |
|---|---|---------------------|------------------------------------|----------------------------|----------------------|
| 1. Pull Chain                                       | In room   | Yes                 | 1                                  | N/A*                       | N/A*                 |
| 2. Wall Control                                     | In room   | Yes                 | Standard: 1- 20<br>HVLS: Unlimited | Fixed: 3<br>Variable: by % | Yes                  |
| 3. Remote<br>Control<br>(Receiver)                  | In Room, Out of<br>Room (*depends<br>on room size,<br>remote range) | Yes                 | Standard: 1-12<br>HVLS: 1-24       | Fixed: 6<br>Variable: by % | Yes                  |
| 4. Smartphone<br>Device                             | In Room, Out of<br>Room   | Yes                 | HVLS: Unlimited                    | 6                          | Yes                  |
| 5. Built-In<br>Occupancy<br>Sensor (Smart-<br>Fans) | In Room, Out of<br>Room   | No                  | 7                                  | 7                          | Yes                  |

\*As pull chains cannot control multiple ceiling fans, speed and lighting control options for multiple fan control do not apply.

Sources: Big Ass Solutions, n.d. (b), Hunter Fan, 2015., Big Ass Solutions, 2013 (b)., Hunter Fan, n.d., EnviroFan, n.d. (a), Marley Engineered Products, 2016, Gescan, 2013., Maxim Integrated, 2011., Big Ass Solutions, 2016 (b)., Home Depot, 2015., The Gadget Reviews, 2016., Y-Lighting, n.d., Home Depot, n.d. (a), Hunter Fan, n.d., Hunter Fan, 2013., SkyBlade Fans, 2017 (b)., SkyBlade Fans, n.d. (b), Hunter Fan Store, n.d., Duke, 2002, Big Ass Solutions, 2017. Lutron, 2017. Crestron, 2017. Hunter Fan, 2017, SkyBlade Fans, 2017 (a)

3 - Design

As evidenced in the previous page, ceiling fan controls are limited in their scope, or in the number of fans they are able to control simultaneously. This limitation can best be explained by (a) electrical equipment load capacity and (b) practical design considerations, both of which are discussed below.

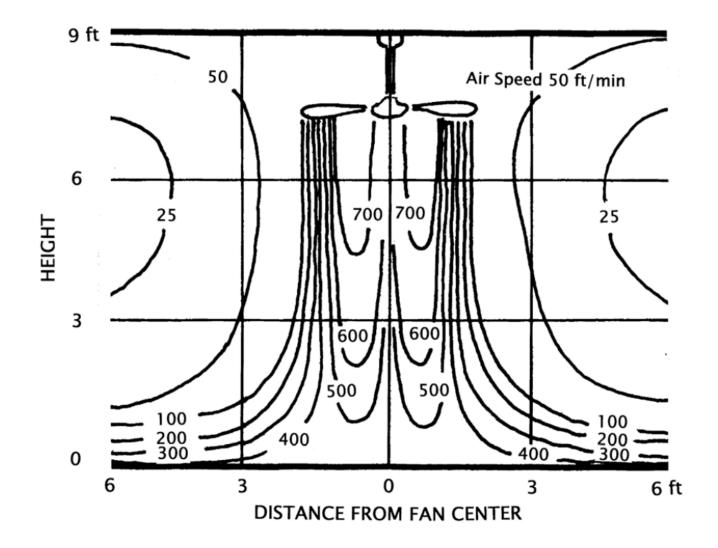
(a) As mandated by National Electric Code standards, a circuit breaker, which is an electrical switch that protects an electrical circuit from damage by over-current should not carry more than 80% of its rated current. Thus, electrical equipment load such as a ceiling fan's load, or amperage size is limited to 80% of the corresponding circuit breaker's size. Therefore, a standard 15-20 amperage circuit breaker will only allow for a 12 amperage load, or about a 12 fan linkage (1 fan ~ 1 amperage). This example excludes other potential appliances such as fan light fixtures that would also exist and draw power from the same circuit. When control technologies are able to control more than 12 fans. as is seen for standard fans and HVLS fans on the previous page, it is due to either smaller load capacities of the fans (smaller diameters, motors) or daisy-chaining fans together, as is commonly done for HVLS fans. HVLS fans are often daisy-chained together, meaning that a master fan is linked to a control device and all remaining fans operate via a Variable Frequency Drive (VFD) command that is linked to the master fan. Daisy-chaining thus allows for a larger load bigger than 12 fans to be linked together, which explains why HVLS fan control devices are able to link unlimited amounts of fans together.

(b) To design for practicality, a cap in the number of ceiling fans that can be controlled by a single control technology will arise. This cap results from the diversity of desired air movement within a space, which limits the practicality of operating multiple fans simultaneously. For instance, in large industrial spaces, demand for multiple fan control often does not exceed 20 to 30 fan linkages as building managers recognize that fan speeds need to be accommodated to the specific space and to the needs of nearby occupants. Therefore, speed settings may vary from slower speeds that produce little to no wind and destratify air to create an even temperature gradient to faster speed settings when a stronger breeze is preferred. Thus, unless a smart-sensor is deployed that contains occupant preferences and environmental conditions, the number of fans controlled by one control device will be constrained.

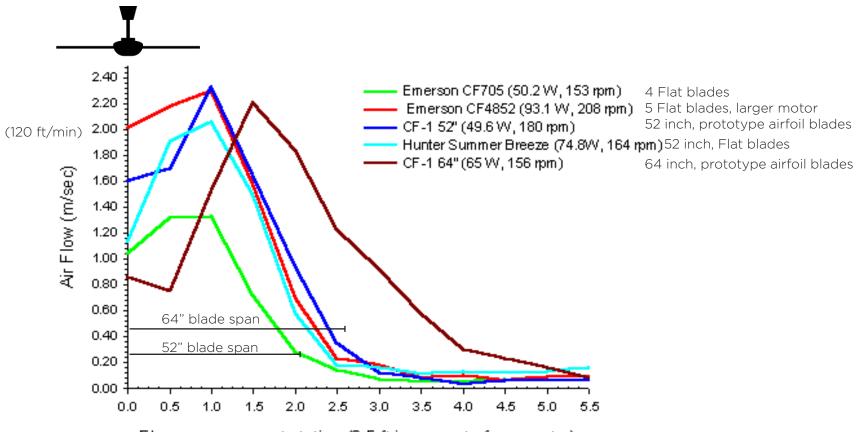
In theory, the listed control limitations would cover all control issues. In practice, however, further issues may arise that are specific to certain fan or control devices. These are to be dealt with on a case by case basis with the designated fan or control manufacturer.

Sources: Home Depot, n.d (a), Duke, 2002. Big Ass Solutions, 2017, Lutron, 2017, Crestron, 2017, Hunter Fan, 2017, MacroAir Fans, 2017

A typical air movement profile of a flat blade fan operating in the summer that is positioned 9 feet off the floor is seen below. Understanding ceiling fan air movement is important in sizing and placing ceiling fans in a space.



Source: Aynsley, 2008 Image Source: Aynsley, 2008 The air movement profile of a ceiling fan varies with fan blade design and size. The figure below shows the air speed profile from the center of 5 different ceiling fans measured at 43 inches off the floor. Three of the below ceiling fans have flat blades, while two have airfoil blades.

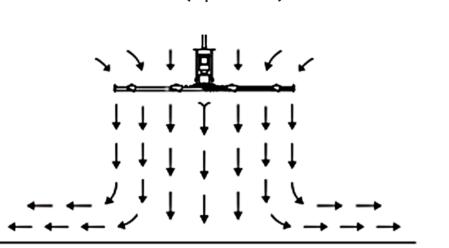


Flow measurement station (0.5 ft increments from center)

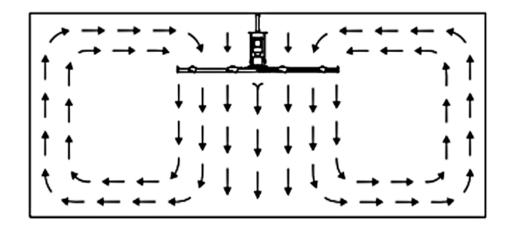
Source: Parker et al, 1999 Image Source: Parker et al, 1999 The typical air movement distribution for an HVLS fan in an open and enclosed area is pictured below. Understanding the air movement profile of an HVLS fan allows designers and installers to better understand a ceiling fan's airflow distribution within a room. The airflow profile of an HVLS fan in an open area can be seen on the left. In this scenario, air moves from the fan blades to the floor and is deflected outward upon reaching the floor. This deflection off the

Airflow (Open Area)

floor is called a "floor jet." On the other hand, the airflow profile of an HVLS fan in an enclosed area, pictured on the right, begins with air radiating outward until it hits a wall, which deflects the air upward above the fan blades. As the area above the ceiling fan is a low-pressure zone, air moves into it until it is then pulled down, which creates a convective current and allows the air movement to sustain its circular pattern and generate a cooling effect.





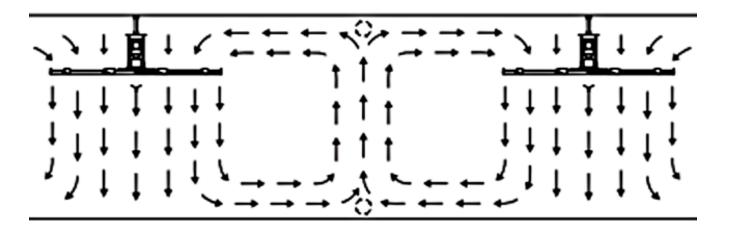


Sources: Big Ass Solutions, 2014 (b), Big Ass Solutions, 2017 Image Source: Big Ass Solutions, 2014 (b)

THINK 27

The typical airflow pattern when multiple HVLS fans are placed in a room are shown below. When HVLS fans are appropriately spaced apart, the performance of each fan can be enhanced by the presence of an adjacent fan. Initially, airflow of multiple HVLS fans is directed downward and it eventually flows outward until it reaches a pressure zone, which is formed when the two fan's air movement meets. The pressure zones behave like walls, as air similarly rebounds off and is pulled upward to the low-pressure zone area above the fan. Usually, the performance of a single HVLS fan improves when working simultaneously with other fans because the pressure zones that form ensure that air is bounded, which causes air to be targeted to a narrower area. This narrow area would not be as adequately served if no nearby fan existed as circulating air would simply continue to disperse outward until it hit the nearest wall.

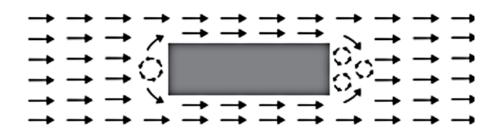
Airflow (Multiple Fans)



Sources: Big Ass Solutions, 2014 (b). Big Ass Solutions, 2017 Image Source: Big Ass Solutions, 2014 (b) Airflow distribution patterns of an HVLS fan with floor obstructions that are either streamlined (thin) or blunt (wide) are provided below. Floor obstructions affect air movement and alter the reach and velocity of air circulating to certain areas within a room. Specifically, for thin or streamlined obstructions, air streaming horizontally is blocked and diverted to pass smoothly around its outside boundaries, with only small bits of air stagnating behind. On the other hand, wide or blunt obstructions change the direction of airflow, causing a larger amount of stagnating air to remain behind the obstruction. Although obstructions are often overlooked in the planning process, the air movement profile with obstructions should be considered when placing or installing HVLS fans as intended cooling or destratification may be undesirably affected. Further, no specific dimensions qualify an obstruction as either streamlined or blunt, and rather what determines the obstruction type is whether the body of the obstruction is parallel or perpendicular to the horizontal air movement.

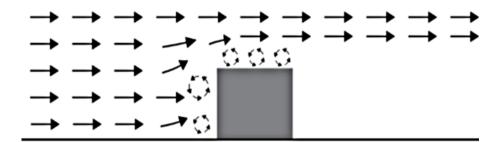
#### Airflow (Obstruction)





Thin, streamlined floor obstructions minimally impact air movement

Blunt Obstruction - Side View

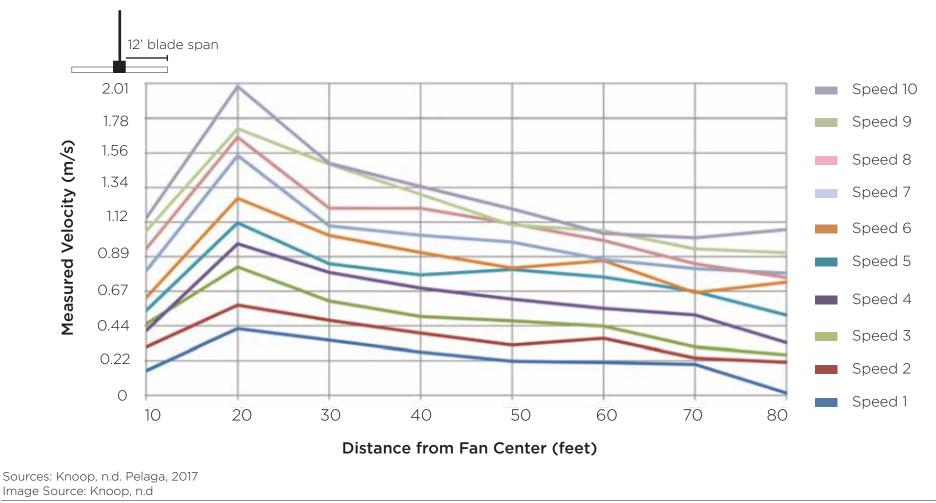


Wide, blunt floor obstructions significantly impact air movement

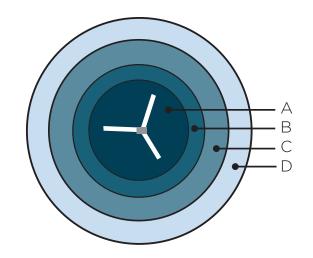
Sources: Big Ass Solutions, 2014 (b), Entrematic Fans, 2016 (b), Big Ass Solutions, 2017 Image Source: Big Ass Solutions, 2014 (b)



The figure below shows the air speed profile for a 24 foot diameter HVLS fan from the center of the fan at 0 feet to 80 feet away from the fan. Airflow readings were taken at four different heights: 4", 24", 43", 67" and were then combined to determine the combined average air velocity at each radial distance. Ten speeds are shown, with the maximum speed (Speed 10) being 58 RPM. As can be seen below, the maximum velocity is achieved slightly outside the fan perimeter, and this is because as moving air is deflected off the floor, it accelerates initially and then slows as it moves further from the fan's center. Since currently no standardized models exist on air speed profile over space for different blade types, the figure below is simply intended to be an initial diagram by which to better understand HVLS fan air movement speed profile over space.



Placement of an HVLS fan is dependent on the fan's area of influence, or the maximum area that the fan can effectively circulate air. Typically, the larger the fan, the larger the area of influence. The below image depicts an HVLS ceiling fan and the corresponding areas of influence A, B, C, D which are organized by the fan's average air velocity within a given area. The accompanying chart details the average air velocity, perceived cooling and diameters of coverage per area of influence. Although this information differs by fan model and manufacturer, the below information is intended to provide general insight on the cooling abilities and coverage capacities of HVLS ceiling fans.



Diameters of Coverage (ft) \_

| Area of<br>Influence | Average Air<br>Velocity (m/s) | Perceived<br>Cooling (°F) | 8 ft Fan<br>Diameter | 16 ft Fan<br>Diameter | 24 ft Fan<br>Diameter |
|----------------------|-------------------------------|---------------------------|----------------------|-----------------------|-----------------------|
| A                    | 2.8 m/s                       | 11-14 °F                  | 8 ft                 | 16 ft                 | 32 ft                 |
| B                    | 1.3 m/s                       | 5-9 °F                    | 15 ft                | 40 ft                 | 50 ft                 |
| С                    | 0.9 m/s                       | 6-8 °F                    | 20 ft                | 60 ft                 | 120 ft                |
| D                    | 0.5 m/s                       | 4-5 °F                    | 128 ft               | 182 ft                | 190 ft                |

Computational Fluid Dynamics (CFD) is a modeling technique that helps predict fluid flow by using mathematical modeling, numerical methods and software tools. Architects and engineers often use CFD to simulate airflow phenomena and thermal distribution in a space to better inform room and building design.

CFD modeling can predict several thermal-fluid phenomena, which can be found below. The model output allows designers and architects to better evaluate and understand the interaction between indoor environmental quality, building envelope, ventilation system performance and thermal comfort.

- (1) Simultaneous Heat Flows
  - (e.g. heat conduction within building enclosures via velocity, temperature measurements)
- (2) Phase Changes
  - (e.g. condensation and evaporation)
- (3) Chemical Reactions (e.g. combustion)
- (4) Mechanical Movements
  - (e.g. fan and occupant movement)

CFD modeling has evolved into an important computational tool for several industries. The increased use of CFD modeling in design and analysis has generated demand for standardized accuracy standards or guidelines to ensure quality control of the models' output. Currently, other engineering professional organizations such as the American Society of Mechanical Engineers (ASME) and the American Institute of Aeronautics and Astronautics (AIAA) provide guidance on performing CFD simulations and validating the results. Efforts have occurred within ASHRAE to set forth guidelines on the use and validation of CFD models which has led to the development of a CFD manual, however no formal, comprehensive standard is currently available.

The accuracy of a CFD model strongly depends on the proper setting of boundary conditions and simulation parameters. Currently, and as a result of the complexity of CFD codes and the expansive range of applications, no robust accuracy guidelines and/or protocols govern CFD modeling. This lack of streamlined has led to diverse settings in CFD simulations and disagreement among professionals on exact accuracy procedures. However, the primary way to currently assess accuracy in CFD models is by means of validation and verification (V&V) procedures. Validation is the process of assessing how well computational results compare to experimental data, or real-life observations. Verification is the process of identifying and quantifying errors in the model. V&V procedures are used to test the model's accuracy, which may be affected by errors and inaccuracies that arise from imprecise input data, limited computer power and gaps in scientific knowledge.

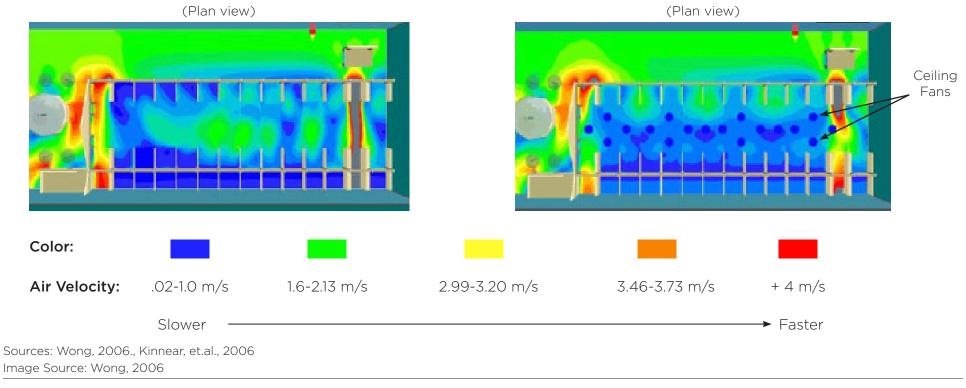
To summarize, while independent V&V procedures are used for quality assurance of CFD modeling, no over arching guideline currently exists that details best practices and standardized verification methods for CFD and CFD codes.

Sources: Gungor, 2015. Srebric, et. al, 2002, NASA, 2008. Chen, n.d. Gungor, 2015, Franke, et. al. 2007, Judkoff, et.al. 2006, Begg, 2012

Wong et. al (2006) perfomed a study in Singapore and ran CFD simulations to investigate the effectiveness of different mechanical ventilation systems in reducing thermal discomfort in dining centers. As ceiling fans were considered, this study serves as an example as to how CFD modeling can be used to understand the impact of ceiling fans on indoor environmental conditions. Provided below is an example of air velocity contour plots obtained with CFD modeling for both a no-ceiling fan scenario and a ceiling fan scenario. The colors in the velocity contour plots represent the velocity magnitude in the studied space, with warmer colors, such as red representing faster air velocity and colder colors, such as blue representing slower air velocity. In using CFD modeling to understand air velocity or air temperature distributions, designers and planners can make more informed decisions on room and building design.

## Air Velocity Distribution (No Ceiling Fan)





HNEI Ceiling Fan Study - Del 2

Applicable ceiling fan codes and standards can be found under equipment testing or air movement as it relates to thermal comfort.

## Energy

Energy Policy and Conservation Act of 1975 (EPCA): defined and established design standards for ceiling fans. The EPCA for ceiling fans was updated in 2017 to require more stringent efficiency standards effective 2020.

## Testing

ANSI/AMCA 230-07: Laboratory Methods of Testing Air Circulating Fans for Rating and Certification. 2007

10 CFR part 430, subpart B, appendix U: *Uniform Test Method for Measuring the Energy Consumption of Ceiling Fans.* (2017)

## Thermal Comfort

ANSI/ASHRAE 55: Thermal Environmental Conditions for Human Occupancy. 2013

EN ISO 7730: Ergonomics of the thermal environment - Analytical determination and interpretation of thermal comfort using calculation of the PMV and PPD indices and local thermal comfort criteria. 2005

EN 15251: Indoor environmental parameters for design and assessment of energy performance of buildings - addressing indoor air quality, thermal environment, lighting and acoustics. 2007

## Upcoming

ASHRAE Standards Project Committee 216: Standard method of test for individual fans (public comment anticipated end 2017)

3 - Design

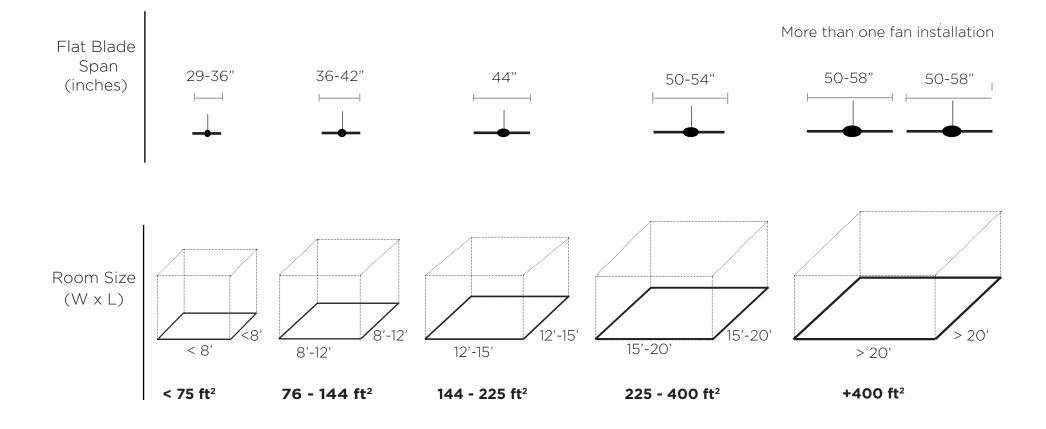


4

# Sizing and Placement

| Flat Blade Sizing and Placement           | 38 |
|---|----|
| Flat Blade Obstruction Guidelines         | 42 |
| Airfoil HVLS Blade Sizing and Placement   | 45 |
| Airfoil HVLS Blade Obstruction Guidelines | 49 |
| Lighting Obstructions: Strobe Effect      | 51 |

To appropriately determine a ceiling fan's size, a measurement of the room's size (width, length) or area (square footage) is required. With this information, optimal flat blade fan size can be decided upon. General industry guidelines exist that provide recommended flat blade span diameters for different room sizes. When installing fans, however, it is common to down-size from the guidelines so that only occupied space is covered. The figure below gives guidance on ceiling fan flat blade span (size) in conjunction with total room size.

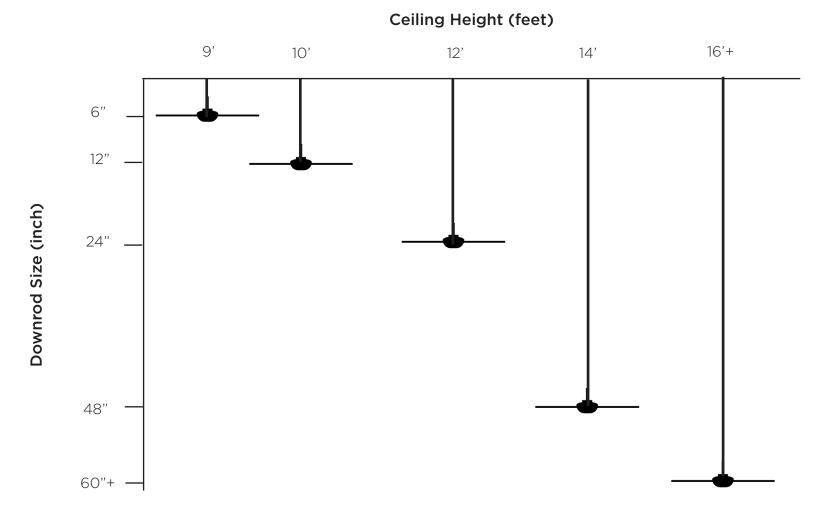


Sources: Bermudez, 2016 (a). Bermudez, 2013 (a). CeilingFan.com, n.d. (a) Hawaii State Energy Office, n.d

Placement of a ceiling fan is determined by ceiling height and room size. The industry standard for the minimum clearance between a fan's blades and the ceiling is 8 inches. Additionally, and as recommended by manufacturers and the International Association of Certified Home Inspectors (InterACHI), fan blades of residential ceiling fans are required to be at least 7 feet from the floor in order to prevent accidental contact with blades. In commercial spaces, fan blades are required to be 10 feet from the building floor. For smaller rooms with ceiling heights of less than 8 feet, low-profile ceiling fans with no downrods are recommended. For larger rooms, with ceiling heights greater than 9 feet, downrods are to be installed so that the fan does not sit too far from the floor and inadequately cool occupants. As shown below, downrod size guidelines for flat blade ceiling fans vary by ceiling height.

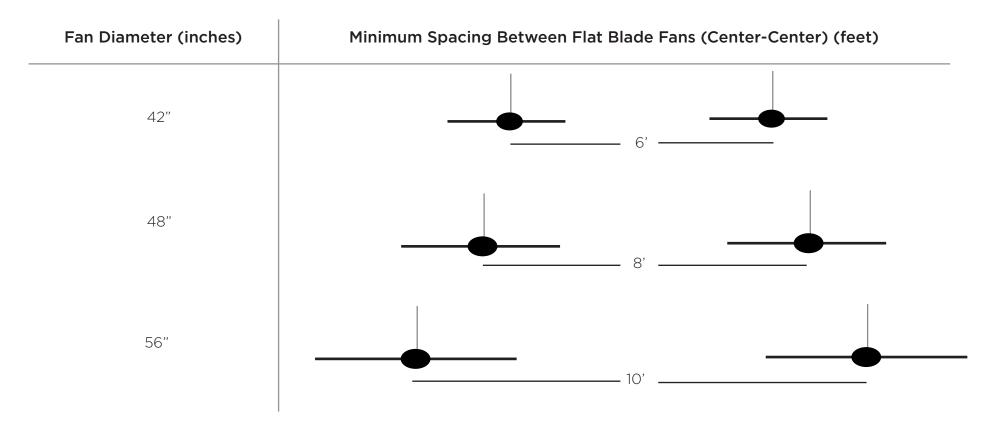
| Ceiling Height  | Recommendation   |      |
|-----------------|--|------|
| Less than 8'    | Choose a low-profile, hugger ceiling fan with no<br>downrod, whose motor housing mounts directly<br>to the ceiling                             |      |
| Greater than 9' | Choose a ceiling fan downrod according to<br>the ceiling height. The next page contains the<br>downrod guidelines for specific ceiling heights |      |
|                 | Dowr   | nrod |

Sources: Gromicko, n.d., Stephens, n.d., CeilingFan.com, n.d. (a), Bermudez, 2016 (a), Bermudez, 2016 (b) Image Sources: Home Depot, n.d. (d), Lumens, n.d As is discussed in the previous page, when a ceiling height exceeds 9 feet, a downrod should be installed on the flat blade ceiling fan in order to provide adequate air circulation for occupants below. The below graph shows the recommended downrod sizes for different ceiling heights. As can be observed, the taller a ceiling is, the larger a downrod is recommended to be.



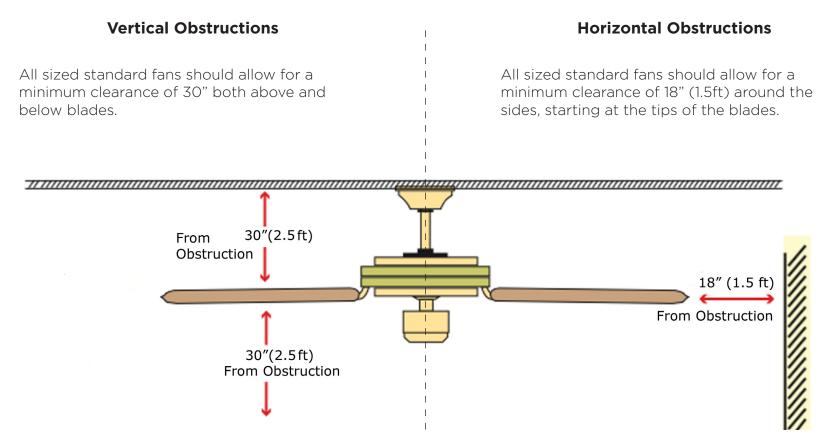
Source: Ceiling Fan.com, n.d (a)

Multiple flat blade ceiling fans may be used to distribute larger volumes of air throughout a room. Currently, no standardized placement guidelines exist for multiple ceiling fans by fan size or diameter. Generally, industry recommends placing fans equidistantly from all vertical walls and these industry recommendations were used to calculate the approximate minimum spacing requirements (in feet) between two fans as shown. Center-to-center spacing refers to the measurement taken from the center of one fan to the center of another neighboring fan.





Obstructions on the ceiling (light fixtures, fire sprinklers, pipes) as well as on the floor (walls, cabinets, floor equipment) have a direct impact on air circulation and distribution since air bounces off objects, causing turbulence. The guidelines below provide information on the maximum allowable distance recommended from flat blade ceiling fans and objects above and below (vertical) as well as to the side (horizontal). Since obstruction guidelines differ by fan model and manufacturer, provided below are the conservative values found in industry publications and specification documents.



Sources: Hunter Fan, 2002; Westinghouse, n.d., Casablanca, 2014 Image Source: Westinghouse, n.d



Minimum distance guidelines for specific obstructions, such as fire-sprinklers and lighting are provided below for flat blade ceiling fans. While a formal association, the National Fire Protection Agency (NFPA) determines placement guidelines of ceiling fans with respect to fire-sprinklers, no such formalized standards exist for lighting, and the provided lighting guideline listed below is acquired from industry articles that provide common guides and tips.

**Fire Sprinklers:** The National Fire Protection Agency (NFPA) creates standards regarding fire sprinkler systems, coined NFPA 13 and 13R. These standards include minimum distance requirements from fire-sprinklers to ceiling fans, which have been developed through testing and evaluation within the NFPA.

- Standard 13 requires residential pendent sprinklers to be located at least 3 feet from the center of the fan. Residential sidewalls are to be located at least 5 feet from the center of the fan.
- Standard 13 states that blades of ceiling fans are not considered obstructions when the surface area of the fan blades encompass less than 50% of the total fan's area from a plan view. Thus, from a plan view, the non-fan area or the open area needs to be at least 50% open for ceiling fans to no longer be considered obstructions for fire-sprinklers. See example (1) on the next page for further specification.
- Standard 13 requires sprinklers to be positioned away from obstructions (such as ceiling fans) at a minimum distance of four times the maximum dimension of the obstruction (ceiling fan). See example (2) on the next page for further specification.

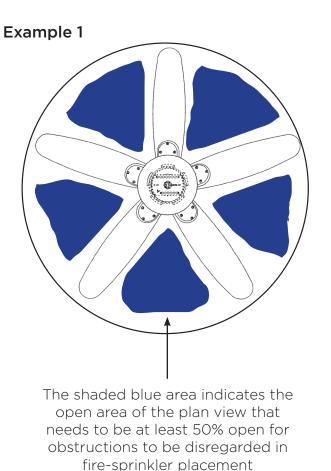
Sources: NFPA, 2016., Faninfl, 2017., City of Colorado Spring, n.d., Ceiling Fan Select, 2017

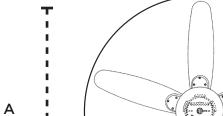
**Lighting:** Lighting fixtures are potential obstructions whose placement can conflict with a ceiling fan. Currently, the minimum clearance recommended between a ceiling fan's blades and light fixtures is 39 inches.

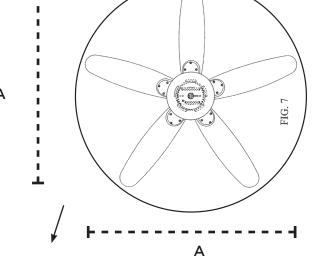


Below are visual representations of the fire-sprinkler obstruction standards outlined in the previous page. Example 1 refers to the standard qualifying ceiling fans as obstructions, while Example 2 refers to distance guidelines from obstructions, such as ceiling fans.

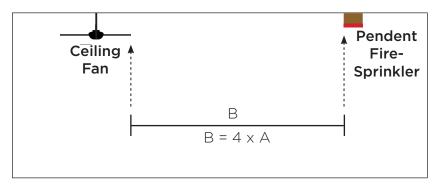
Example 2





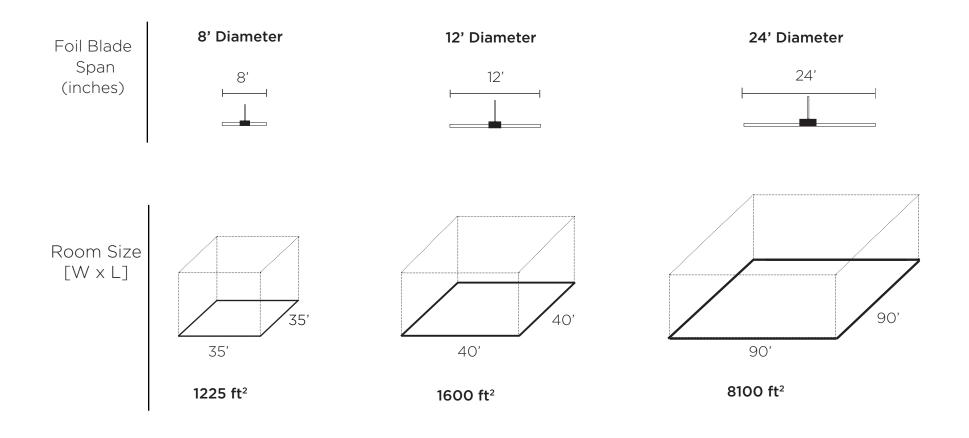


Minimum Distance Guidelines:



Sources: NFPA, 2014. City of Colorado Spring, n.d. DSPS, n.d Image Source: Google, 2011

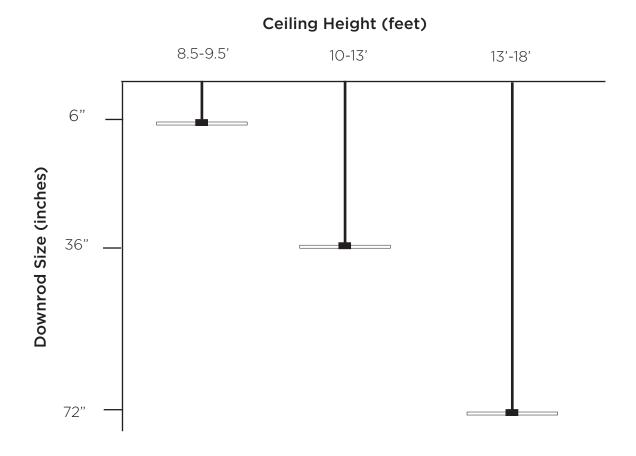
Standard sizing guidelines for HVLS fans with airfoil blade diameters of 8, 12 and 24 feet are shown below. These guidelines are developed by determining the area of the room (length, width) as well as the ceiling height and then assessing the amount of space each fan can effectively cool. These sizing guidelines are obtained from manufacturer specifications, therefore they do not apply to all airfoil blade HVLS fans and are solely meant to provide the general idea of airfoil blade HVLS fan sizing. To understand how ceiling height affects fan placement, see next page.



Sources: Whalenado, 2015, SkyBlade, n.d. OSHA, n.d. Blue Giant, 2017

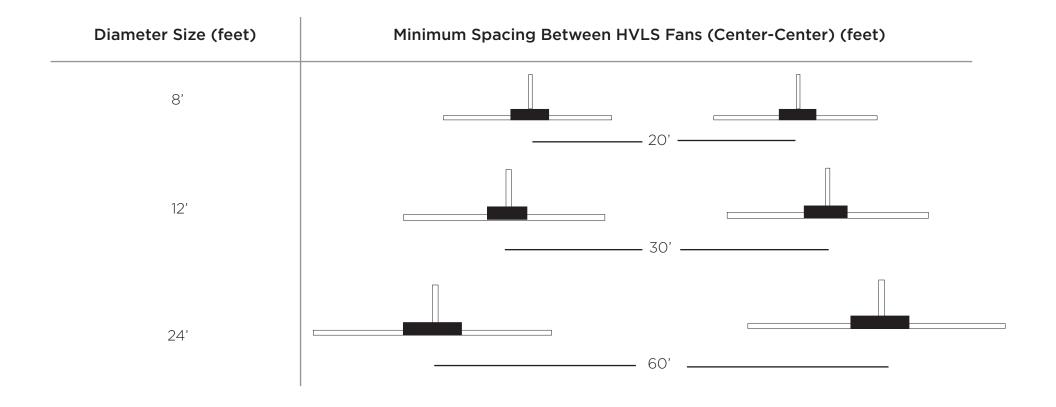


Similar to flat blade ceiling fan placement, the placement of an airfoil blade HVLS fan is dependent on ceiling height. For commercial spaces, in which HVLS fans are commonly used, fan blades should be located at least 10 feet above the floor. As commercial and industrial spaces often have tall ceilings, downrods are used to lower a fan and optimize airflow or destratification. Just as was the case for flat blade ceiling fan placement, downrod size is proportional to a ceiling's height. As no comprehensive and standardized HVLS fan placement guideline currently exists, the recommended placement guidelines shown below are obtained from manufacturer documentation, and therefore should only be consulted as a preliminary guideline when making placement decisions.



Sources: Gromicko, n.d. MyFan, n.d

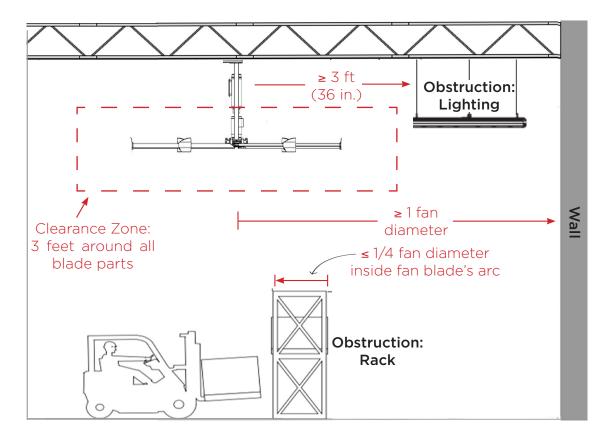
In large industrial spaces or facilities, the use of multiple HVLS ceiling fans can more effectively distribute air throughout all desired areas. To ensure optimal performance and minimize overlap of coverage, spacing the fans properly and equidistantly is of vital importance. Below are spacing guidelines by ceiling fan size (in feet) that are based on the general rule that HVLS fans should be spaced at a center-to-center distance that is at least 2.5 times the fan's diameter. Center-to-center spacing refers to the measurement taken from the center of one fan to the center of another neighboring fan.



Sources: Hansen Wholesale, n.d (b), Big Ass Solutions, 2014 (b)



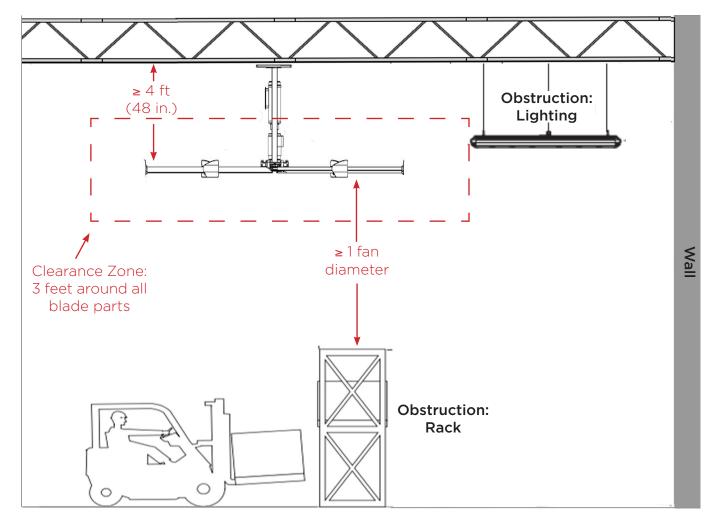
Large commercial and industrial spaces typically require HVLS ceiling fans to circulate large volumes of air on large floor spaces. In these vast spaces, obstructions can exist to the side as well as below the ceiling fan. To properly and safely place HVLS ceiling fans so that they do not conflict with occupants, neighboring equipment, fire-sprinklers or lights, placement guidelines that recommend minimum clearances have been developed, and those that are of horizontal orientation are seen below. The below clearance guidelines are explained in further detail in following pages. It is important to note that obstruction guidelines are design and manufacturer specific, and so the distances listed below and on the next page are the conservative guidelines found in the industry.



## **Horizontal Obstruction Guidelines**

Sources: Big Ass Solutions, 2014 (b), Big Ass Solutions, 2016 (b), Entrematic Fans, 2016 (a) Entrematic Fans, 2016 (b) Image Source: Big Ass Solutions. 2016 (b)





## Vertical Obstruction Guidelines

Sources: Big Ass Solutions, 2014 (b), Big Ass Solutions, 2016 (b), Entrematic Fans, 2016 (a), Entrematic Fans, 2016 (b) Image Source: Big Ass Solutions, 2016 (b)

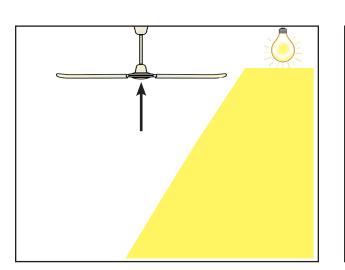


| General<br>Guidelines:                      | <ul> <li>All sized HVLS fan blade parts should allow for a minimum clearance distance of 3 feet (36") from obstructions such as light fixtures, cables, fire sprinklers and other building components.</li> <li>All sized HVLS fans should allow for a minimum clearance distance of 4 feet (48") between the fan blades and the ceiling.</li> <li>All sized HVLS fans should allow for a distance of 1 fan diameter from the center of the fan to any wall.</li> </ul>  |
|---|--|
| Guidelines<br>for Specific<br>Obstructions: | <ul> <li>Fire Sprinklers: The NFPA 13, 2013 Edition provides a set of design and installation rules based on performed testing. They are listed below:</li> <li>HVLS fans need to be centered between four adjacent sprinklers.</li> <li>HVLS fan blades need to be positioned at least 3 feet away from the sprinkler deflector so that the blades do not interfere with sprinkler activation.</li> <li>HVLS fans nuct be all programmed to shut down immediately upon 90 seconds of activation of a water flow signal from the fire alarm system. Fans must also shut off after the activation of smoke and heat detection devices.</li> <li>Lighting: As noted in the general guidelines above, when a ceiling fan is installed into a room with existing light fixtures and/or panels, a minimum clearance of 36 inches (3 feet) is recommended. Placing a ceiling fan too close to lighting can cause a strobe effect, where light perpetually dims and brightens as it penetrates through a moving fan. To minimize this effect, it is important to maximize the horizontal and vertical distance between the blade and the light panel. When a complete separation is not possible, it is best to minimize the light and fan overlap, as the frequency of the strobe effect increases when the fan and light grow closer.</li> <li>HVAC: For HVLS ceiling fans situated near HVAC equipment should allow for a minimum clearance distance that is equal than or equal to 1 fan diameter.</li> <li>HVLS fans located abow HVAC equipment should allow for a minimum clearance distance that is greater than or equal to 2 fan diameters.</li> <li>Equipment (Racks, Walls, etc) Below: For HVLS ceiling fans installed above solid equipment, such as racks, walls or storage, the fan must allow for a minimum clearance distance that is greater than or equal to 2 fan diameters.</li> <li>HVLS fans located above solid equipment must allow for a minimum vertical clearance greater than or equal to 2 fan diameters.</li> <li>HVLS fans located above solid equipment must allow for a minimum vertical clearance</li></ul> |

Sources: MacroAir Fans, 2016. Krell, 2016, Klaus Bruckner, 2011, Allianz, 2012



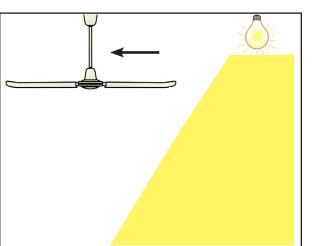
Ceiling fans are often installed in rooms with existing light fixtures and if the fan is placed too closely to a light panel or fixture, a strobe or flicker effect may occur. A strobe effect is when light perpetually brightens and dims as it penetrates and passes through a moving ceiling fan, causing dizziness and confusion for occupants. While no exact distance guidelines exist to minimize potential strobe effects with standards of HVLS ceiling fans, industry recommends that the horizontal and vertical distance between the light and the fan be maximized between the light fixture and the ceiling fan. Additionally, if a complete separation between the fan and the light panel is not possible, it is best to minimize overlap since the frequency of the strobe effect increases with the amount of overlap. When the light panel is outside of the perimeter of the fan, three recommendations exist: (1) the fan should be moved up far enough so that light does not penetrate through, which can be done by shortening the downrod (2) the fan should be moved away from the light so that light does not penetrate through, or (3) the light's angle of dispersion or the field angle should be reduced so that light does not penetrate through.

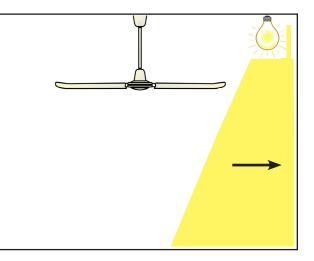


**Recommendation #1:** 

Move Ceiling Fan Up

**Recommendation #2:** Move Ceiling Fan Away From Light Recommendation #3: Reduce Light's Angle of Dispersion





Sources: MacroAir Fans, 2016, EM Reno Blog, 2012 Image Sources: EM Reno Blog, 2012, Radiant, n.d.



# Operations

5

| Energy              | 54 |
|---------------------|----|
| Performance Metrics | 56 |
| Thermal Comfort     | 60 |
| Air Movement        | 62 |
| Alliesthesia        | 67 |

Ceiling fans are often compared to air conditioning (AC) as both technologies aim to increase occupant comfort but through different mechanisms and at different building scales. Ceiling fans provide a perception of cooling by increasing air movement while AC provides cooling by actually altering the air temperature and humidity. Another key difference is that ceiling fans and window AC units operate on a room level, while one central AC system can cool an entire building. Therefore, many window AC and ceiling fans are required to achieve the same coverage as central AC.

Cooling an indoor space, using a central AC, a window AC or a ceiling fan has different power requirements. For example, a typical 3 ton central AC used to cool a 1600 feet<sup>2</sup> residential application would consume 3,000 watts per hour while a window AC unit would consume 1200 watts per hour. An average ceiling fan consumes 60 watts per hour or 2% of the central AC system and 5% of the window AC units. In other words, 50 ceiling fans would be required to achieve the same power draw as a central AC unit and 20 ceiling fans would be required to achieve the equivalent power draw of a window AC unit. Thus, with air conditioning installed in almost two-thirds of

homes, ceiling fans can provide significant energy savings by reducing AC operating hours, retrofitting AC systems to be smaller or eliminating them altogether.

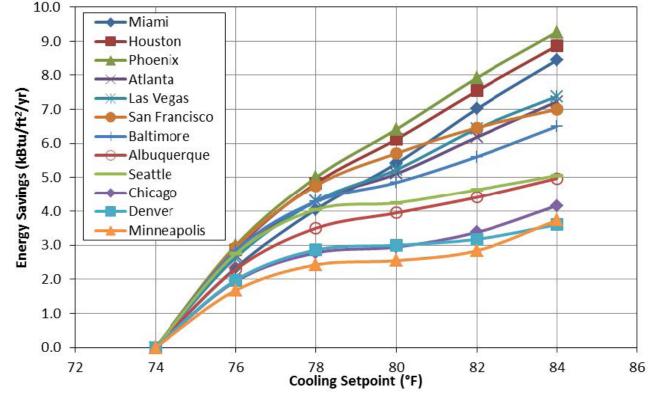
The large range in power consumption between AC and ceiling fans translates into a significant cost differential. "The Energy Information Administration has found that fewer than 4 percent of households with central air turn it off during the workday when no one is home1". As an edge example, the energy required to run a window AC in Hawaii all year round would cost \$2,716 to operate while a ceiling fan's would cost \$175.

|                         | Avg Power<br>(Watts) |        | Average Annual<br>Cost in HI <sup>4</sup> |
|-------------------------|----------------------|--------|---|
| Central Air Conditioner | 3,000                | \$0.78 | \$6,833                                   |
| Window Air Conditioner  | 1,200                | \$0.31 | \$2,716                                   |
| Ceiling Fan             | 60                   | \$0.02 | \$175                                     |

Sources: 1. Tortorello 2011. Schauer, 2012; In Table: 3. Energy Information Administration, 2017. 4: 8760 hours in a year

Ceiling fans do not save energy directly but rather indirectly through HVAC energy reduction. Air movement provided from ceiling fans keeps occupants comfortable even though HVAC setpoints are relaxed. The National Renewable Energy Laboratory used energy models to estimate total energy savings in medium office building by incrementally changing setpoints from 74°F to 84°F in 12 U.S. cities. Moving the cooling setpoint from 74°F to 84°F resulted in a 3-9% total energy savings as seen below. Cities in hotter climates (e.g. Phoenix, Houston, Miami) had larger total energy savings.

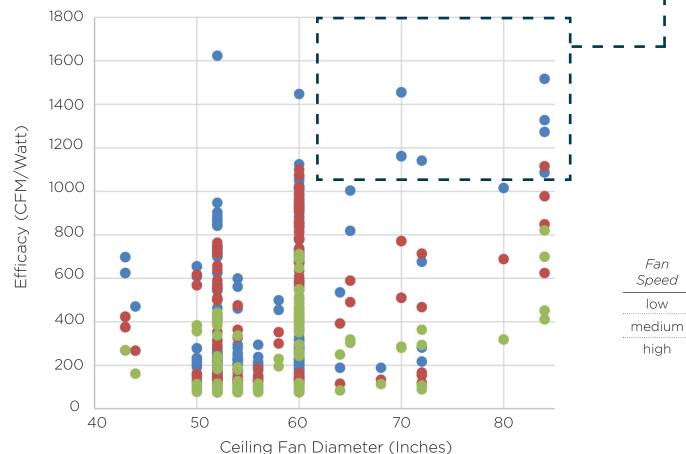
# Key Takeaway: Relaxing cooling setpoints by 10°F can save 3-9% in total energy use



Source: Kiatreungwattana 2016 Image Source: Kiatreungwattana 2016

5 - Operations

ENERGY STAR has promoted energy efficiency of ceiling fans by providing minimum airflow and efficacy requirements, with efficacy measured as CFM/ Watt. Currently there are over 100 available models (14% of market) meeting those requirements. The following chart shows the range of efficacies for ENERGY STAR ceiling fans based on ceiling fan diameter. Efficacy is measured on three fan speeds (i.e., low, medium, high). The conclusions are that lower fan speeds and larger ceiling fan diameters have higher efficacies.



# Key Takeaway: Lower fan speeds and larger diameters have higher efficiencies

| Fan<br>Speed | Minimum<br>Airflow | Minimum Efficacy<br>Requirement |
|--------------|--------------------|---------------------------------|
| low          | 1,250 CFM          | 155 CFM/watt                    |
| medium       | 3,000 CFM          | 100 CFM/watt                    |
| high         | 5,000 CFM          | 75 CFM/watt                     |

Source: ENERGY STAR, 2016 Image Source: ENERGY STAR, 2016



Life Cycle Cost (LCC) and simple payback calculations are good cost-benefit tools for evaluating alternatives. As part of DOE's analysis to amend the ceiling fan energy conservation standard for more stringency (effective March 2020), they conducted LCC analyses and simple payback calculations to understand the economic impacts to consumers of higher efficiency ceiling fans. More specifically, the initial cost of a ceiling fan with higher efficiency will likely increase but consumers will benefit from a reduction in operational costs.

In both LCC and simple payback calculations. lower values are preferred. Mathematically, LCC calculations add the total customer expense (i.e. installation, first cost, sales tax) and discounts future operational costs (i.e. maintenance, repair, operations) across the product's lifespan to time of purchase. A simple payback calculation gives the amount of time, in years, before a consumer breaks even. Simple pavback is the ratio of incremental change in initial investment from a high efficiency product to a baseline divided by the change in first year operating expenses.

Key factors and results from the LCC and Simple Payback calculations used by the DOE are summarized on the subsequent pages and will likely be useful for future cost benefit analysis.

## Factors Average Value

Sales Tax7.16%Operating HoursStandard Fans: 6.45 hrs/day; Large-diameter fans: 12 hrs/dayProduct LifetimeMean: 13.8 years; Median: 13 yearsDiscount RateResidential: 5% for replacement, 3% for new installation; Commercial/Industrial: 5%

5 - Operations



Key LCC and payback findings from the average standard fan and large diameter fan product types conducted by the DOE are organized below. Four efficiency levels above the baseline are included to show the range of results within each product type. For all efficiency levels, standard fans and large diameter fans have simple paybacks of less than 5 years with lower efficacy levels paying back in less than 1 year. The equations used to calculate Life Cycle Cost and Simple Payback are provided on the following page. The equations are based on Standard Fans with Efficiency Level 4 (EL4).

## Standard Fans (blade span of 44,52 & 60 inches)

| Efficiency<br>Level (EL) | -   | -        | Avg First Year's<br>Operating Cost<br>(2015\$) | Avg Lifetime<br>Operating Cost<br>(2015\$) | Avg Life Cycle<br>Cost (2015\$) | Simple Payback<br>(years) |
|--------------------------|-----|----------|--|--|---------------------------------|---------------------------|
| Baseline <sup>1</sup>    | 44  | \$113.49 | \$16.99  | \$190.29                                   | \$303.79                        |                           |
| 1                        | 58  | \$113.49 | \$12.75  | \$144.06                                   | \$257.55                        | <1 yr                     |
| 2                        | 64  | \$113.49 | \$11.48  | \$130.20                                   | \$243.70                        | <1 yr                     |
| 3                        | 72  | \$124.95 | \$10.33  | \$117.58                                   | \$242.53                        | 1.7 yr                    |
| 4                        | 110 | \$158.01 | \$5.86   | \$75.92                                    | \$233.93                        | 4 yr                      |

<sup>1</sup>baseline: = flat, wood blades with an AC motor and pull chain control sizes

## Large Diameter Fans (blade span of 8,12 & 20 feet)

| Efficiency<br>Level (EL) | -   | -         | Avg First Year's<br>Operating Cost<br>(2015\$) | Avg Lifetime<br>Operating Cost<br>(2015\$) | Avg Life Cycle<br>Cost (2015\$) | Simple Payback<br>(years) |
|--------------------------|-----|-----------|--|--|---------------------------------|---------------------------|
| Baseline <sup>2</sup>    | 83  | \$4119.72 | \$292.21                                       | \$2921.38                                  | \$7041.10                       |                           |
| 1                        | 92  | \$4119.72 | \$262.99                                       | \$2632.90                                  | \$6752.62                       | <1 yr                     |
| 2                        | 101 | \$4261.44 | \$239.08                                       | \$2396.87                                  | \$6658.31                       | 2.7 yr                    |
| 3                        | 115 | \$4458.32 | \$210.14                                       | \$2110.93                                  | \$6569.25                       | 4.1 yr                    |
| 4                        | 156 | \$4706.71 | \$156.42                                       | \$1624.11                                  | \$6330.82                       | 4.3 yr                    |

<sup>2</sup> baseline = curved aluminium blades and 3-phase induction motors

Source: U.S. Department of Energy, 2016 (c)

To provide information on the method that the DOE used to arrive at Average Life Cycle Cost (LCC) and Simple Payback estimates, equations and example calculations performed for a Standard Fan at an efficiency level of 4 (EL 4) are provided below.

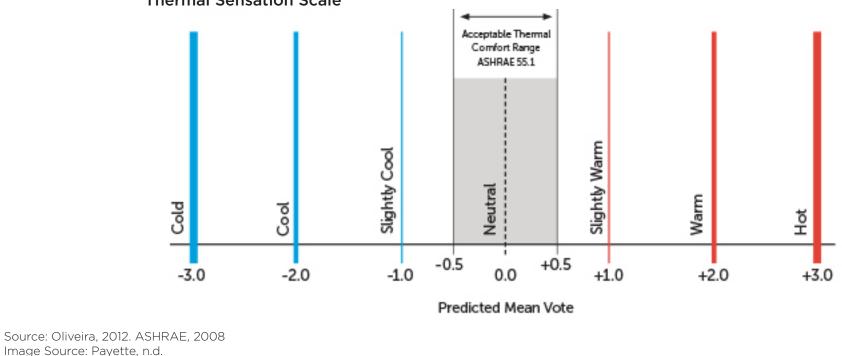
| <b>Life Cycle Cost Equation:</b> LCC = $ C + \sum_{t=1}^{N} OC_t / (1+r)^t$  | Simple Payback Equation: PBP = $\triangle$ IC / $\triangle$ OC  |
|--|---|
| Average Life Cycle Cost<br>Equation*:Avg LCC = Avg IC + Avg OCIC: Total installed cost (in dollars)<br>∑: Summation over a lifetime, from year 0 to year N<br>N: Lifetime of product (in years)<br>OC: Operating cost (in dollars)<br>r: Discount rate<br>t: Year for which the operating cost is being determined | PBP: Payback period (in years)<br>IC: Increase in the total installed cost between the more<br>efficient equipment (EL 1, 2, 3, 4) and the baseline efficiency<br>equipment<br>OC: Decrease in first-year annual operating cost |
| Example Calculation:<br>Average Life Cycle Cost (Standard Fans, EL 4)  | Example Calculation:<br>Simple Payback Equation (Standard Fans, EL 4)   |
| Average Life Cycle Cost<br>Equation: Avg LCC = Avg IC + Avg OC   | Simple Payback Equation: PBP = $\triangle$ IC $/\triangle$ OC   |
| Avg LCC = \$158.01 + \$75.92   | (Avg IC <sup>EL4</sup> - Avg IC <sup>Baseline</sup> )<br>PBP =<br>(Avg FY* OC <sup>EL4</sup> - Avg FY* OC <sup>Baseline</sup> )   |
| Avg LCC = \$233.93   | (\$158.01 - \$113.49)   |
| *As Avg OC includes OC at each year discounted and summed<br>over the lifetime of the product, the above Avg LCC equation is<br>simply summing Avg IC and Avg OC together  | PBP = = 4.04 years<br>(\$5.86 - \$16.99)<br>*FY: First Year   |

Source: U.S. Department of Energy, 2016 (c)



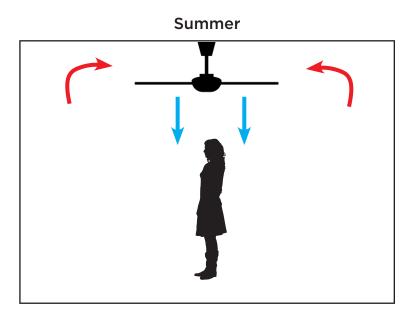
Thermal comfort is defined in the EN (ISO) 7730 Standard as "the mental condition that expresses satisfaction with the surrounding environment". Thermal comfort is assessed using Predicted Mean Vote (PMV), a thermal sensation scale that predicts the average value of occupants' thermal comfort ratings in a given environment.

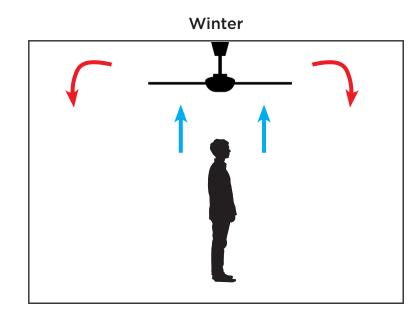
The six main influencing factors involved in thermal comfort are: (1) air temperature, (2) relative humidity, (3) mean radiant temperature, (4) air speed, (5) clothing value, (6) metabolic rate. In surveys, occupant's thermal sensation ratings are recorded using a seven point-scale that ranges from cold to neutral to hot, as is shown below. The acceptable thermal comfort range denoted by ASHRAE-55 Standards falls between -0.5 and +0.5. This range represents 80% thermal occupant acceptability, where 80% of occupants are satisfied with their thermal conditions



**Thermal Sensation Scale** 

Ceiling fans are intended to be used year-round, day and night. In the summer or in warm climates, fans primarily provide perceived cooling by pushing down cooler air and providing increased air movement. In the winter, the fan directs the airflow upward to circulate the warm air that has risen to the ceiling down to the occupant level.



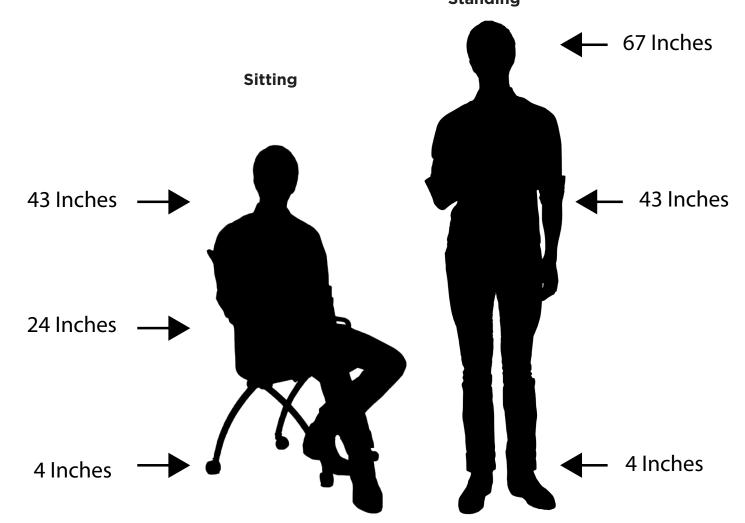


| Perceived<br>Comfort | Cooler   | Warmer   |
|----------------------|--|--|
| Mechanism            | Convection and evaporation   | Air mixing which reduces vertical air stratification   |
| Fan Speed            | Varies (but often high)  | Low  |
| Fan<br>Rotation      | Forward or counterclockwise  | Reverse or clockwise   |
| Energy<br>Impact     | Increase cooling HVAC set-point temperatures<br>(e.g. from 74°F to 78°F) which can save energy<br>by reducing how often the cooling system is on<br>and possibly alter the required size of the HVAC<br>system | Reduce heating HVAC set-point temperatures<br>(e.g. from 68°F to 65°F) which can save energy by<br>reducing how often the heating system is on |

Source: Bermudez, 2013 (b)



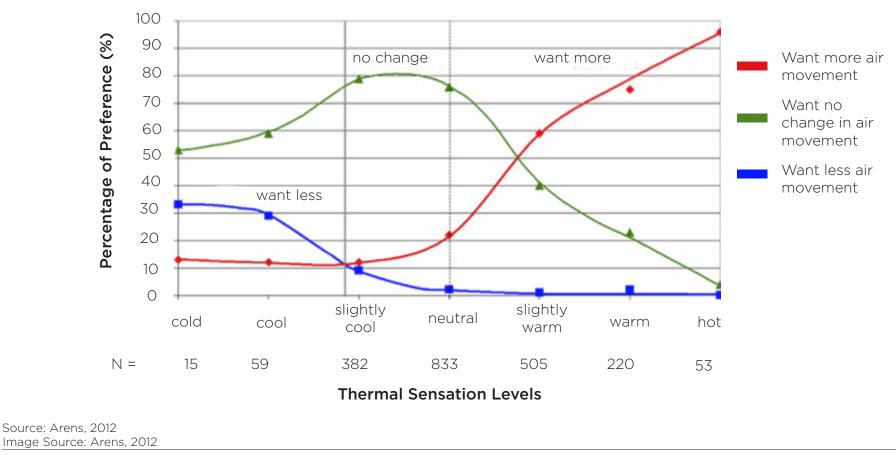
Air temperature and average speed shall be measured at the 4 inch, 24 inch and 43 inch levels for seated occupants and at the 4 inch, 43 inch and 67 inch levels for standing occupants. These variables are measured at 3 different heights because of the stratification in air velocity and temperatures within a space. When calculating thermal comfort, the most extreme observed values should be taken. **Standing** 



5 - Operations

Air speed, or rate of air movement is important for the comfort of occupants in indoor environments. Research has shown that there exists a need for increased air movement, especially when occupant sensation falls within the range of neutral to hot. Arens (2012) created the graphic below to understand air movement preferences by thermal sensation level. This study's findings suggest that when an environment is perceived as either neutral, slightly warm, warm or hot on the thermal sensation scale, there is a clear increasing trend or preference for more air movement.

Key Takeaway: A need for increased air movement exists in warm indoor environments



Air Movement Preference by Thermal Sensation Level

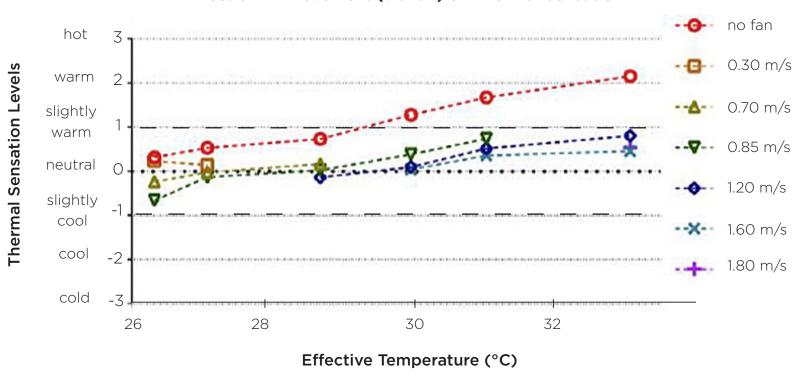
HNEI Ceiling Fan Study - Del 2



5 - Operations

As ceiling fans circulate air and create perceived cooling for occupants in warm indoor conditions, they are solutions for the existing need for more air movement. Tests performed in warm temperature conditions of 28°C and 30°C (82.4°F and 86°F) at all speeds and at 26°C (78.8°F) at high speeds provide evidence that air movement from ceiling fans improves thermal sensation. The figure below from Arens, et al (2013) demonstrates that the use of ceiling fans maintains thermal sensation ratings within a comfort range of 1 (slightly warm) to -1 (slightly cool) while the no fan scenario results in higher levels of reported warm sensations at each temperature condition.

Key Takeaway: Ceiling fans can improve thermal sensation ratings in warm temperature conditions



### Effect of Air Movement (via fan) on Thermal Sensation

Source: Arens, et al, 2013, (k) Image Source: Arens, et al, 2013, (k) The air movement direction can differ by the type of ceiling fan used. As opposed to standard ceiling fans positioned perpendicular to the floor, dualhead ceiling fans have two opposing head fixtures that are oriented diagonally. This difference in orientation alters which part of body receives the most air movement. Airflow directed towards occupants from the front, side, back, bottom and top can be used to increase perceived cooling when focused at regions of exposed skin (e.g. face, neck, ankles, and hands). Since clothing choice varies by day and activity, optimizing the room air direction for maximum cooling is likely not practical. Furthermore, room air speed and temperature have a bigger influence on overall comfort.



Sources: Toftum, et al, 1997. Todde, 2000, Arens, 2013 (g), Arens, 2013 (h) Image Source: Toftum, et al., 1997



Airflow intermittency has the possibility of increasing comfort for occupants. Intermittency can be separated into either speed or fan head oscillation. Academic studies have compared constant speed, fixed fans to speed pulse and x-axis oscillation fans, which are each defined below. There is no consensus regarding whether constant or oscillating airflow provides better thermal comfort for occupants. The table below contains findings from the experiments.

Key Takeaway: Constant fans are preferred to x-axis oscillation fans and speed-pulsing fans are preferred to constant fans, however since these three configurations have not been tested together, no definitive conclusions can be made on which configuration is best for optimizing thermal comfort

Constant: Airflow is occurring continuously in one direction and one speed setting over a period of timeSpeed Pulse: Airflow is focused in one direction and cycles through multiple speed settingsX-Axis Oscillation: Airflow remains at one speed setting and the fan head moves horizontally

| Article Name   | Experiment Configuration   | Year | Constant | Speed<br>Pulse | X-Axis<br>Oscill. | Findings   |
|--|--|------|----------|----------------|-------------------|--|
| Enabling<br>Energy-efficient<br>Approaches to<br>Thermal Comfort<br>Using Room Air<br>Motion                               | <ul> <li>16 college student subjects</li> <li>11 different fan settings and<br/>air direction configurations</li> <li>Surveyed rating thermal<br/>comfort and thermal<br/>sensation</li> </ul> | 2014 | Х        |                | X                 | <ul> <li>No statistical differences<br/>between the 'no-fan' and X-axis<br/>oscillating fan configurations</li> <li>Statistical significance found<br/>between 'no-fan' and constant<br/>fan configurations</li> </ul> |
| Effects of Air<br>Temperature,<br>Humidity, and<br>Air Movement on<br>Thermal Comfort<br>under Hot and<br>Humid Conditions | <ul> <li>64 college aged students</li> <li>7 different air movement<br/>patterns: constant, pulse,<br/>random</li> <li>Surveyed rating thermal<br/>sensation</li> </ul>                        | 1994 | X        | X              |                   | • Statistically significant results<br>show pulsing air speeds provides<br>more perceived cooling than<br>constant air velocity  |
| A Study of<br>Occupant Cooling<br>by Personally<br>Controlled Air<br>Movement  | <ul> <li>119 subjects</li> <li>Range of warm<br/>temperatures and air<br/>movements</li> <li>Surveyed preferences<br/>regarding air movement</li> </ul>  | 1998 | Х        |                | ×                 | • Statistically significant results<br>show that constant mode cools<br>better than X-axis oscillation<br>mode at the same mean wind<br>speed  |

Sources: Pasut, et al, 2014. Tanabe, et al, 1994. Arens, et al, 1998, Arens, 2013 (i)

Alliesthesia is the phenomenon that describes pleasure as a result of changes in body temperature. Different types of alliesthesia exist, and those most related to thermal comfort are spatial and thermal alliesthesia, both of which are explained below. With the emergence of new ceiling fan capabilities such as oscillating fans and intermittent speed settings, ceiling fans are increasingly capable of inducing alliesthesia or pleasurable sensastions for occupants within a space.

Thermal alliesthesia describes the sensation that results from a new stimulus such as air movement and a subject's temperature state. For positive thermal alliesthesia to take place, two things need to occur: (1) the body needs to be in a non-neutral state, at minimum a slightly warm or slightly cool state, and (2) the stimulus needs to restore the body in the direction of its neutral state. For instance, if a subject's current temperature is warm or higher than their neutral state temperature, introducing a ceiling fan will cause a positive sensation or a 'pleasant breeze' as it will restore the body to its neutral state. However, if a subject's current temperature is cool, or lower than their neutral state temperature, introducing a ceiling fan will now cause thermal displeasure or an 'unpleasant draft', as it makes the occupant even colder. **Spatial alliesthesia** is the pleasure that arises from differences in temperature across the skin's surface. Just as with thermal alliesthesia, positive sensations occur when the heating or cooling stimulus is in the opposite direction of the initial body state and closer to the neutral state. Recent studies have found that small variations in skin temperature created with localized stimuli can result in positive pleasure sensation, therefore targeting certain body parts simultaneously with personalized heating or cooling systems such as desk or ceiling fans can create pleasurable sensations. An example of spatial alliesthesia is a pleasure that arises from a wind breeze on the face during a hot day.

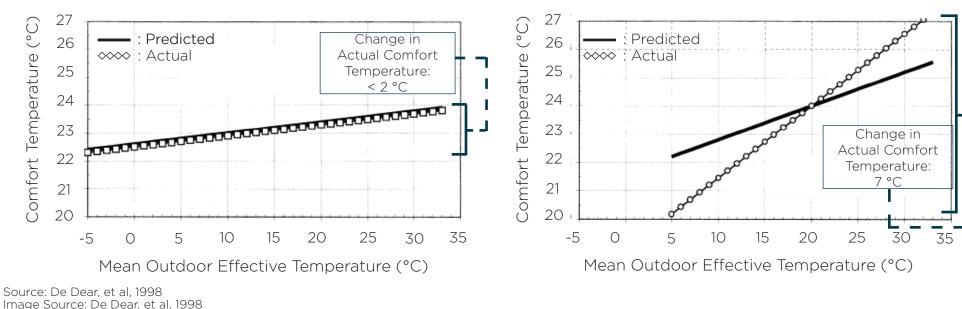
In practice, designing for spatial alliesthesia is more appealing than for thermal alliesthesia as it does not necessitate intentionally creating an uncomfortable situation which is then to be corrected for, and rather is based on assessing the benefits of stimulus directed at specific body parts.



Sources: De Dear, 2015. Parkinson, et al, 2012. De Dear, n.d. Parkinson, et al, 2016 (a). De Dear, 2011. Parkinson, et al, 2014. De Dear, et al, 1998. De Dear, 2009. Brager, et al, 2015

Alliesthesia can occur when a variable stimulus, such as air movement interrupts an occupant's static environment. Ceiling fans provide opportunities for creating dynamic environments as they are capable of providing intermittent air movement. Newer control modes such as the Haiku Whoosh mode automatically alters speeds at different times throughout the day to simulate natural breezes and create variability in air movement. Moreover, findings from a research study (shown below) seeking to understand comfort in fixed or dynamic environments suggests that occupants in naturally ventilated buildings are able to tolerate a wider comfort temperature range than occupants in buildings with centralized. HVAC systems. The likely explanation is that the wider range of adaptive opportunities provided with natural ventilation systems, such as variability and dynamic stimulus from vents and operable windows leads to greater comfort even with warmer temperatures. Therefore, providing intermittent air movement with ceiling fans as a way to replicate natural breezes may increase how often pleasurable sensations are felt, which can improve the overall thermal comfort level of occupants.

Key Takeaway: Occupants in variable and dynamic indoor environments are able to remain comfortable in warmer temperatures



#### **Buildings With Centralized HVAC**

#### **Buildings With Natural Ventilation**

THINK

68



6

# Literature Review

| Introduction                           | 72 |
|--|----|
| Source & Content Diversity             | 73 |
| Bibliography                           | 75 |
| Acronyms                               | 94 |
| Annendix: Electronic Literature Review |    |

Compiled in the following pages is a literature review that contains all the sources and articles read in support of Subtask 1a and Subtask 1b research. The intent of the literature review is to provide an overview and transparency into the types of sources used so others may build off the knowledge assembled.

The literature review is comprised of three main parts. First, the source diversity and content diversity summaries provide a snapshot into the range of sources and topics covered to quickly convey the scale of research. Second, there is an bibliography separated into articles, images and personal conversations with manufacturers and industry professionals. Some of the article references are specifically cited in the report while others were used for context only. Lastly, an acronym list is included for easy referral.

Also, there is a literature review containing a comprehensive summary of all the articles reviewed including a brief abstract and key takeaways from each article. This information is available in a excel and not included in this report due to its size.



Diversity of sources used for Subtask 1a, Market Assessment and Subtask 1b, Technical Assessment research.

Source Type

|   | Unique<br>Sources | Reference<br>Count |
|---|-------------------|--------------------|
| Government<br>[e.g. EPA, DOE, NREL, CARB] | 13                | 18                 |
| Academia<br>[e.g. CBE]                    | 36                | 61                 |
| Manufacturers<br>[e.g. BAF, Del Mar]      | 20                | 50                 |
| Other<br>[e.g. trade organizations]       | 49                | 66                 |
| Total                                     | 118               | 195                |

Diversity of article content used for Subtask 1a, Market Assessment and Subtask 1b, Technical Assessment. Articles can include information across many content areas.

|             |   | Content Category   |                   |                       |                    |        |
|-------------|---|--------------------|-------------------|-----------------------|--------------------|--------|
|             |   | Reference<br>Count | Design<br>Product | Design<br>Application | Thermal<br>Comfort | Energy |
| Source Type | Government<br>[e.g. EPA, DOE, NREL, CARB] | 18                 | 5                 | 9                     | 4                  | 9      |
|             | Academia<br>[e.g. CBE]                    | 61                 | 8                 | 24                    | 49                 | 8      |
|             | Manufacturers<br>[e.g. BAF, Del Mar]      | 50                 | 27                | 28                    | 1                  | 0      |
|             | Other<br>[e.g. trade organizations]       | 66                 | 29                | 31                    | 10                 | 10     |
|             | Total                                     | 195                | 69                | 92                    | 64                 | 27     |

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#### PERSONAL COMMUNICATIONS

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Below contains a list of acronyms relevant to this report.

- AC Alternating Current AC Air Conditioning
- AIAA American Institute of Aeronautics and Astro
- AMCA Air Movement and Control Association
- ANSI American National Standards Institute
- APRISES Asia Pacific Research Initiative for Sustainable Energy Systems
- ASHRAE American Society of Heating Refrigerating and Air-Conditioning Engineers
- ASME American Society of Mechanical Engineers
- BAF Big Ass Fans
- BLDC Brush Less Direct Current
- CARB California Air Resource Board
- CBE Center for the Built Environment
- CFD Computational Fluid Dynamics
- CFM Cubic Feet Per Minute
- CFR Code of Federal Regulations
- DC Direct Current
- DOE Department of Energy

| EPCA       | Energy Policy and Conservation Act          |
|------------|---|
| FPM        | Feet Per Minute                             |
| FSEC       | Florida Solar Energy Center                 |
| HVAC       | Heating Ventilation and Air Conditioning    |
| HVLS       | High-Volume Low-Speed                       |
| InterNACHI | International Association of Certified Home |
|            | Inspectors                                  |
| LCC        | Life Cycle Cost                             |
| NEC        | National Electric Code                      |
| NFPA       | National Fire Protection Agency             |
| PMDC       | Permanent Magnet Direct Current             |
| PMV        | Predicted Mean Vote                         |
| PPD        | Predicted Percentage Dissatisfied           |
| VFD        | Variable Frequency Drive                    |
| V&V        | Validation & Verification                   |





1500 Sansome Street, San Francisco, California 94111 mkthink.com | office@mkthink.com | 415.402.0888